



ROYAL INSTITUTE
OF TECHNOLOGY

COMSOL
CONFERENCE
2018 LAUSANNE

Modelling the Dynamic Viscous and Thermal Dissipation Mechanisms in a Fibrous Porous Material

Brad Semeniuk

Department of Aeronautical and Vehicle Engineering

KTH Royal Institute of Technology – Stockholm, Sweden

Peter Göransson

Department of Aeronautical and Vehicle Engineering

KTH Royal Institute of Technology – Stockholm, Sweden

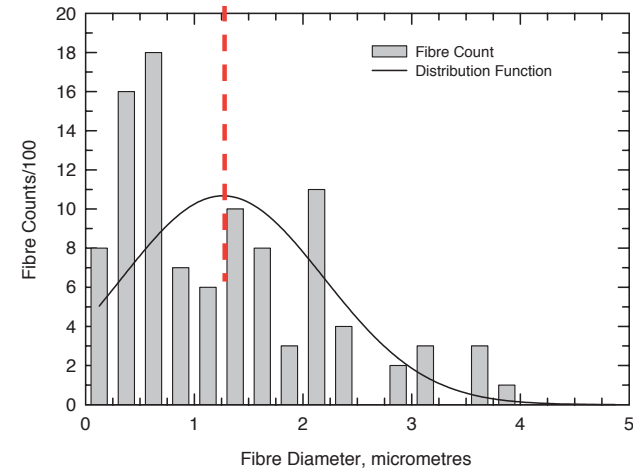
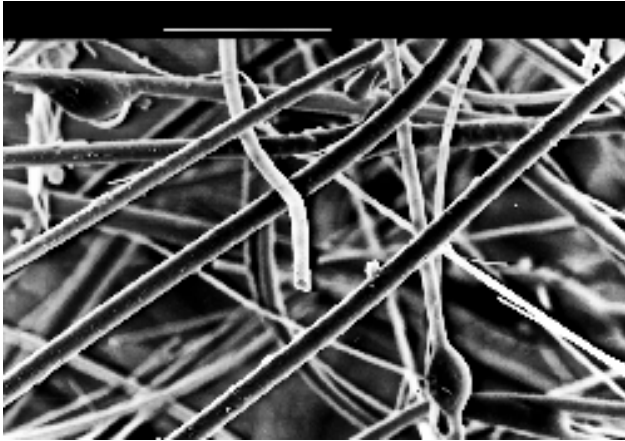


ROYAL INSTITUTE
OF TECHNOLOGY

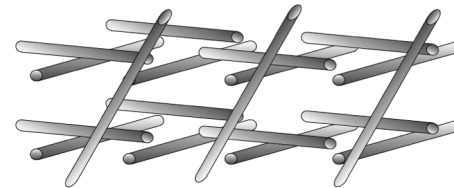
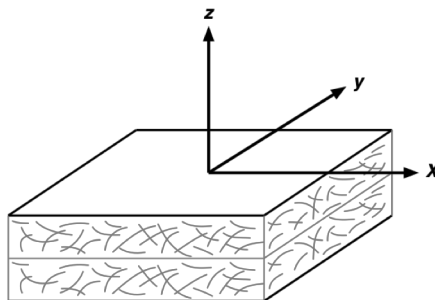
Presentation Outline

- New poroelastic fibre microstructure material model with analytical viscous drag force and heat transfer expressions for cylindrical fibres.
- Based only on geometrical properties of the material, and constitutive properties of the solid fibres and surrounding fluid.
- Includes diameter and fibre orientation distributions, which are tied closely to material production methods.
- Utilize COMSOL thermoviscous acoustic fluid FE models as a *Virtual Laboratory*, to verify the assumption of no viscous and thermal boundary layer interaction between neighbouring fibres.

Transversely Isotropic Fibre Assumption



- 10 kg/m³ bulk density glassfibre aircraft thermal insulation
- 1.28 micron mean fibre diameter



- Transversely isotropic (stacked isotropic layers with some fibres orientated vertically)

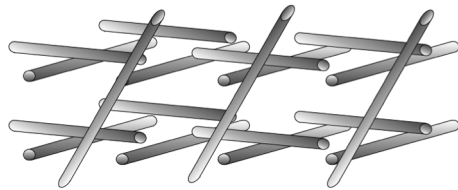
Momentum Equations in Terms of Viscous Drag Forces

Solid Momentum Equations

$$\phi \rho_s \ddot{u}_i = \left(\frac{\partial \sigma_{ii}}{\partial x_i} + \frac{\partial \sigma_{ij}}{\partial x_j} + \frac{\partial \sigma_{ik}}{\partial x_k} \right) - F_{D_i}$$

Fluid Momentum Equations

$$(1 - \phi) \rho_f \ddot{U}_i = \frac{\partial \sigma}{\partial x_i} + F_{D_i}$$



* Viscous Drag Force Impedance Matrix

$$\{\mathbf{F}_D\} = [\bar{\mathbf{Z}}] \{\dot{\mathbf{u}} - \dot{\mathbf{U}}\} \quad \text{relative motion between solid and fluid}$$

$$[\bar{\mathbf{Z}}] = \frac{\phi}{A} \begin{bmatrix} J & 0 & 0 \\ 0 & J & 0 \\ 0 & 0 & K \end{bmatrix} \quad \text{transversely isotropic, isotropic or fully anisotropic}$$

$$Z_l = 2\pi a k_\beta \mu_f \frac{H_1^{(2)}(k_\beta a)}{H_0^{(2)}(k_\beta a)} \quad \text{longitudinal cylinder oscillations}$$

$$Z_t = j\pi \rho_f \omega a^2 \left[1 - \frac{4H_1^{(2)}(k_\beta a)}{k_\beta a H_0^{(2)}(k_\beta a)} \right] \quad \text{transverse cylinder oscillations}$$

* defined in terms of diameter and fibre orientation distributions

Fluid Continuity with Heat Transfer from Fibre to Fluid

Stress-Strain for Transverse Isotropy

$$\{\sigma\} = [D]\{\varepsilon\}$$

Fluid Dilatation modified for Non-Equilibrium
(heat flow from fibres to fluid results in
thermal expansion of the fluid)

$$\chi\sigma = R\varepsilon + (Me_{xx} + Me_{yy} + Qe_{zz})$$

Scaling Coefficient

$$\chi = \left[1 - \frac{\alpha\eta R}{j\omega\rho_f C p_f (1-\phi)} \right]$$

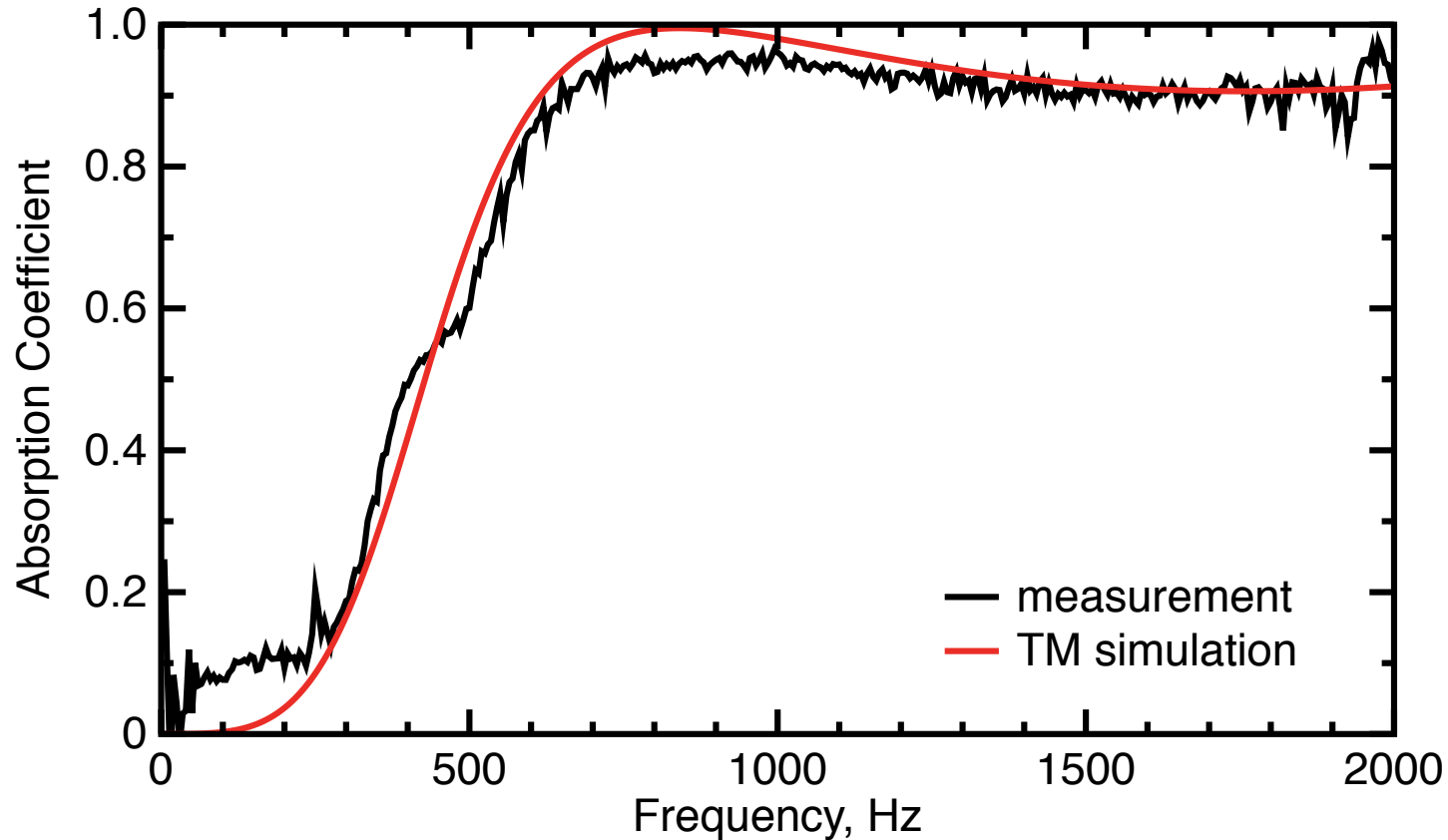
$$\alpha = \eta T'_f \left\{ \frac{\rho_f C p_f (1-\phi)}{\bar{Y}_e} + \frac{1}{j\omega} \right\}^{-1}$$

Thermal Impedance at fibre/fluid
interface

$$\bar{Y}_e = \frac{\phi}{A} \left[\frac{1}{Y_f} + \frac{1}{Y_s} \right]^{-1}$$

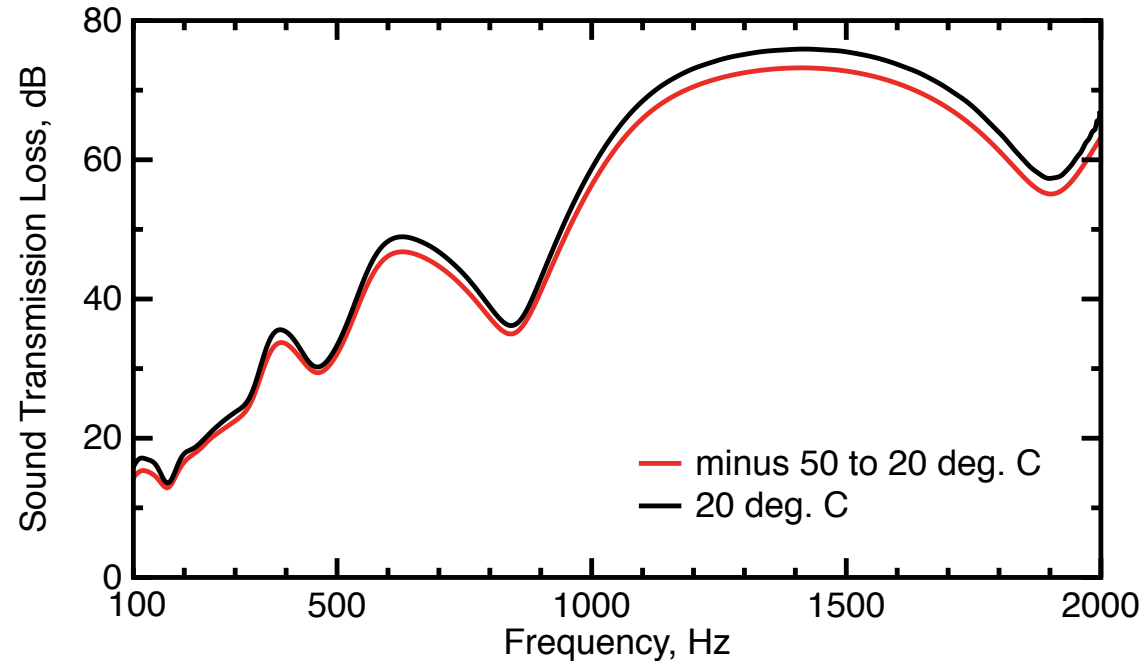
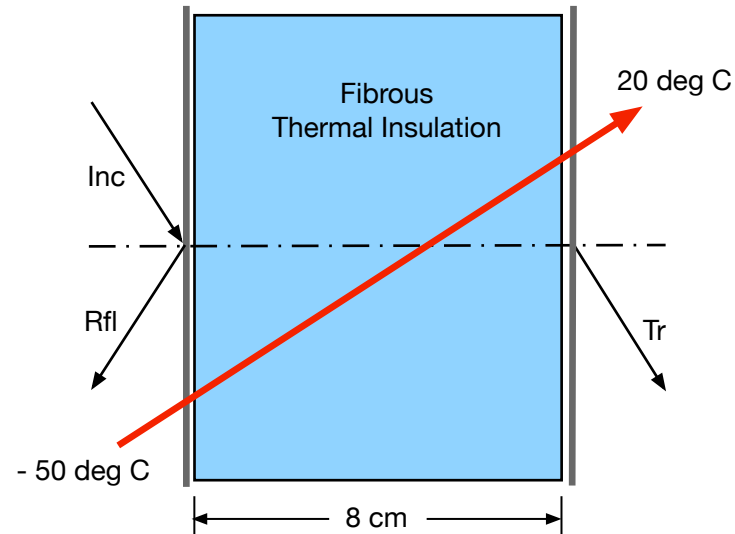
based on oscillatory fluid
and fibre thermal fields

Impedance Tube Sound Absorption



- Measurement: 50 mm sample thickness, rigid backing impedance tube.
- TM Simulation: 1.28 micron mean fibre diameter, assumed 50 deg. fibre inclination angle.

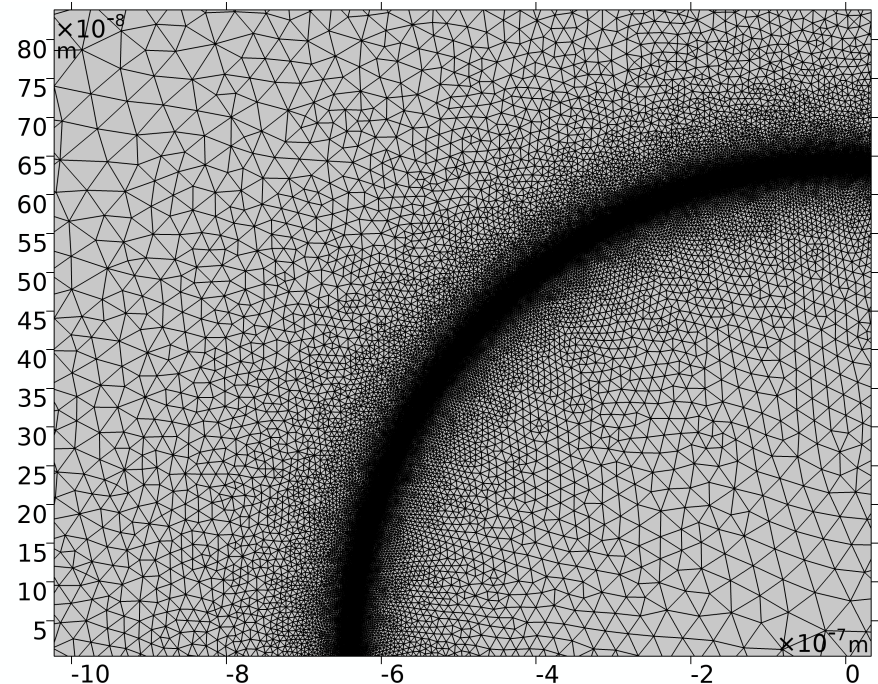
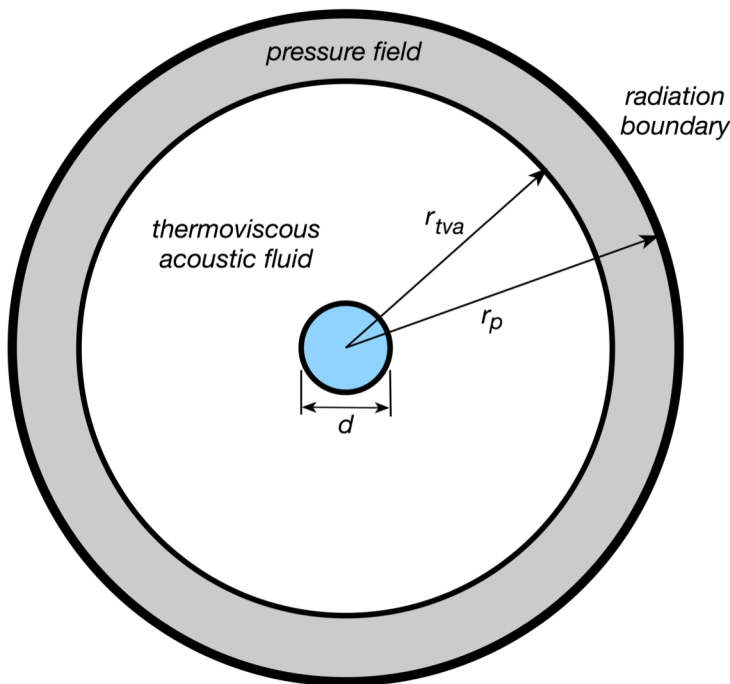
Sound Transmission Through Aircraft Fuselage with Temperature Gradient



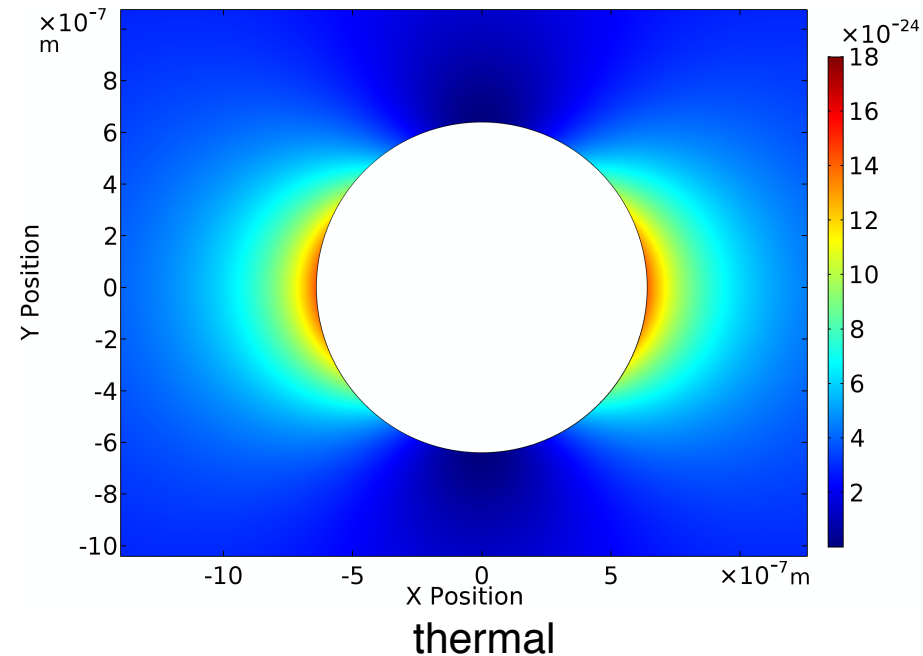
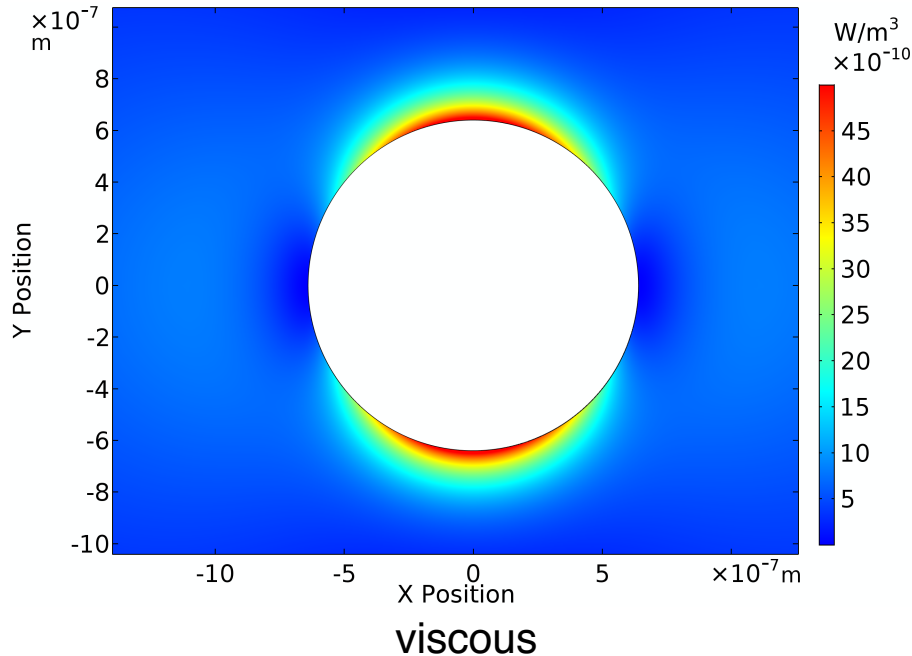
- Diffuse STL through an aircraft fuselage during inflight conditions.
- - 50 to 20 deg. C temperature gradient.
- Traditional room temperature analysis over-estimates acoustic performance.

Thermoviscous Acoustic Fluid Modelling COMSOL as a *Virtual Laboratory*

- 1.28 micron solid cylinder, 0.01 micron transverse oscillations (@ 1 Hz)
- 15 mm thermoviscous fluid, 5 mm acoustic field with external radiation boundary (to ensure complete decay of radiating waves)
- internal stress field of fibre is modelled
- 12 million DOF, nanoscale element sizes along fibre boundary

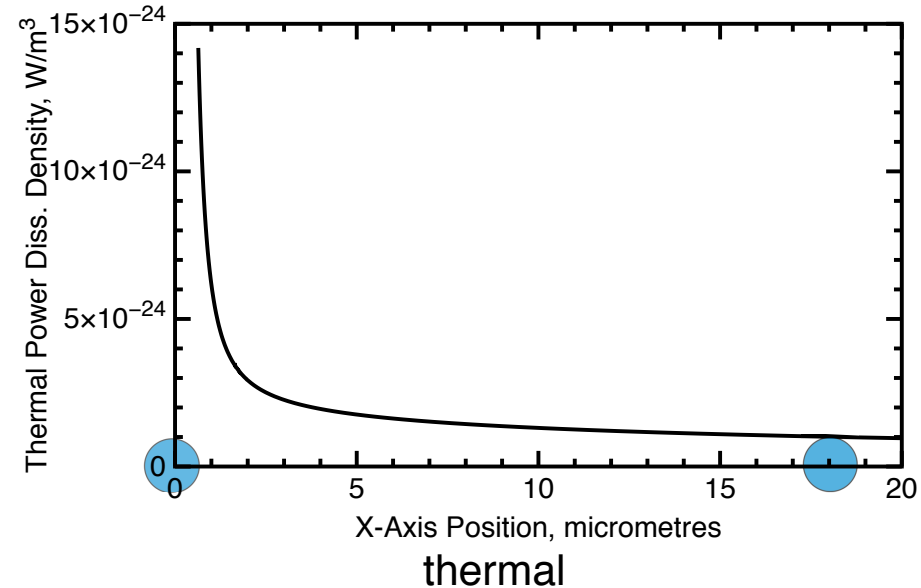
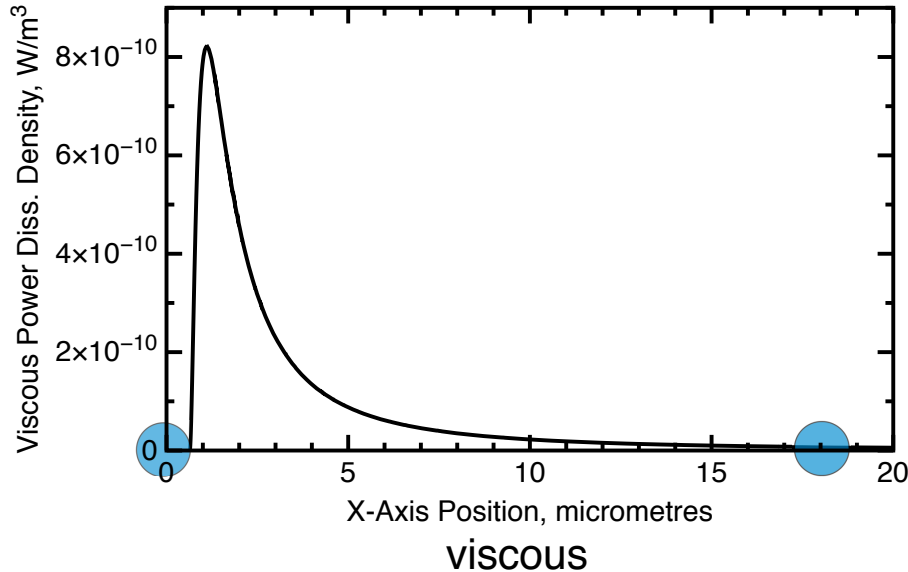


Viscous and Thermal Boundary Layers on a Single Fibre



- Viscous and thermal boundary layers concentrated near fibre surface, at the micro and nano scale.
- Analytical transverse dynamic drag impedance: 87665 Ns/m^4 at 1 Hz.
- FE solution: 87642 Ns/m^4 . **0.02% difference.**

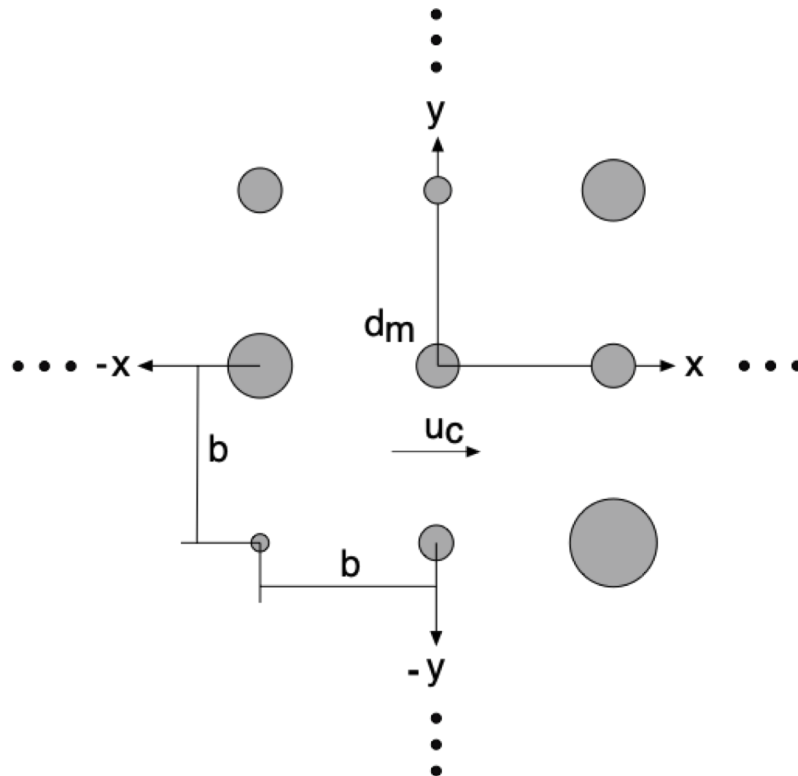
Viscous and Thermal Boundary Layer Strength



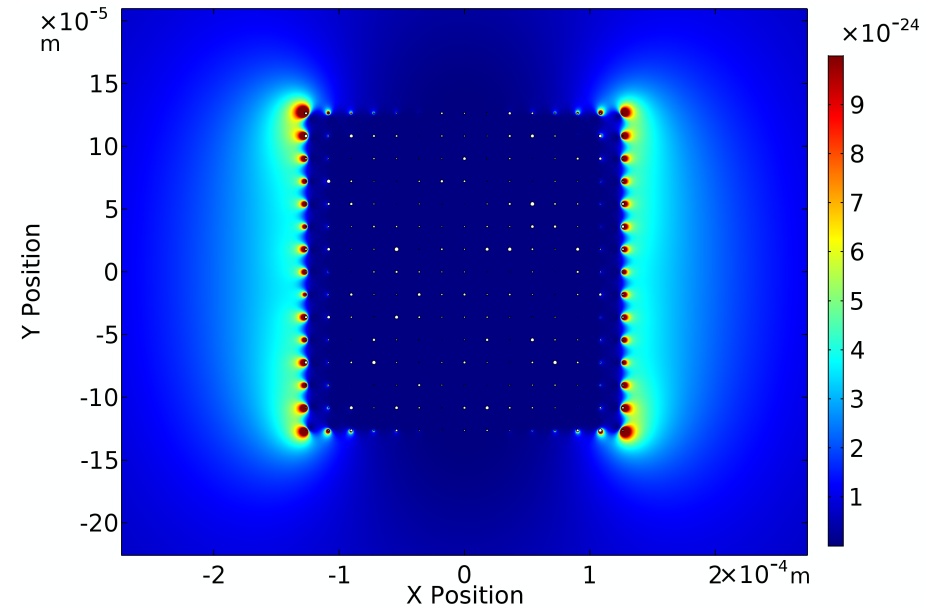
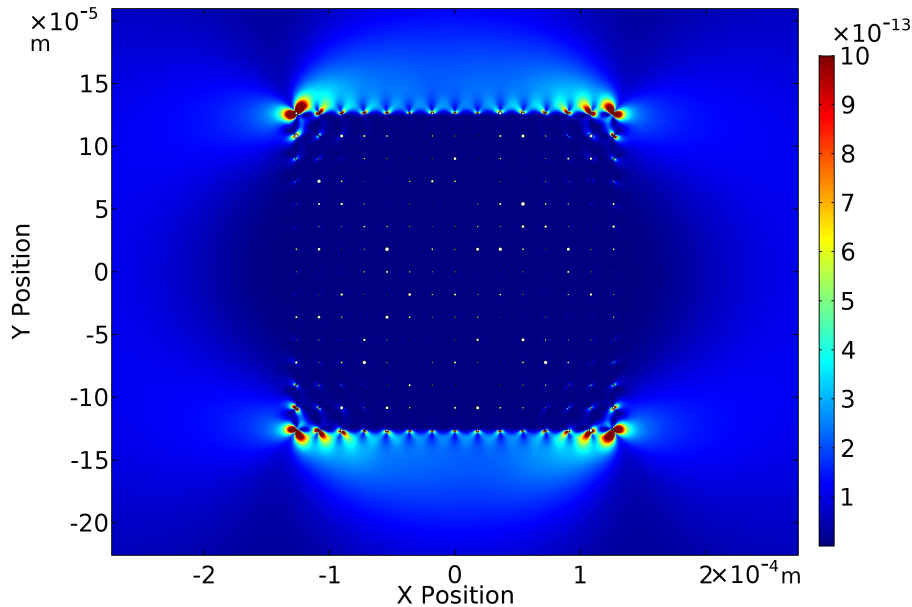
- Viscous dissipation effects dominate in the material.
 - Boundary layer strengths are concentrated near fibre surface.
 - Viscous boundary layer has dissipated 98% at the neighbouring fibre.
- Viscous boundary layer penetration depth is 2.2 mm at 1 Hz. Thermal is 2.6 mm.

Multifibre Array Modelling

- 225 regularly spaced fibre array representative of diameter distribution.
- Allows viscous and thermal boundary layer interaction between fibres to be considered.
- **35 million DOF in model.**

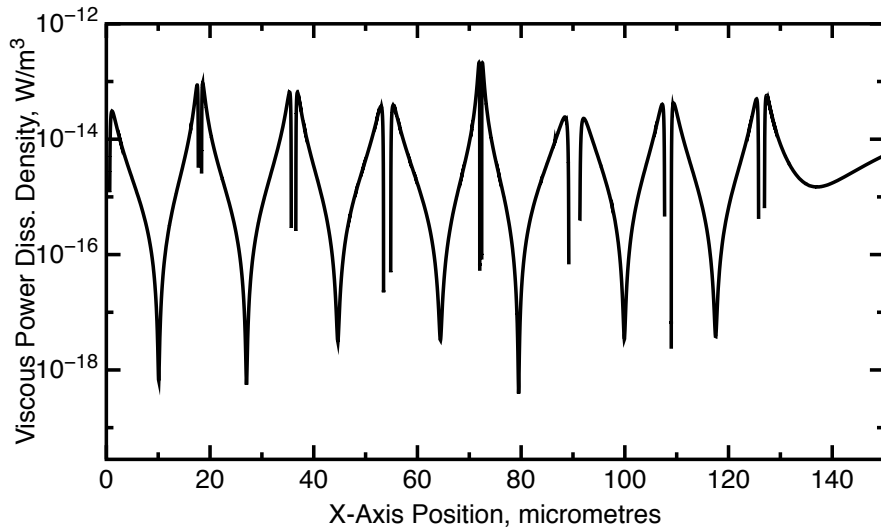


225 Fibre Array: Viscous and Thermal Boundary Layers Interaction

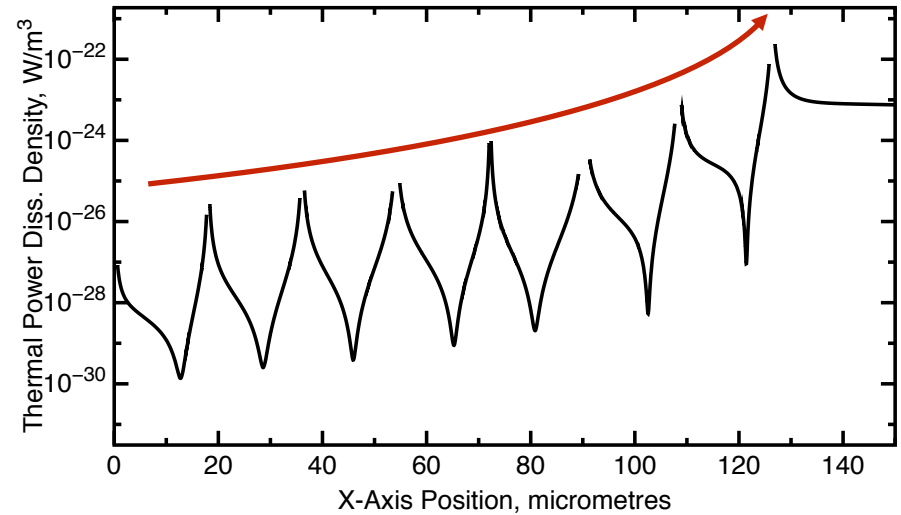


- Viscous drag force estimate within approx. 1.1% of analytical solution.
- Amplitude increases observed for outer thermal boundary fibres.

Viscous and Thermal Boundary Layers Along X-Axis



viscous



thermal

- Uniqueness of viscous boundary layer amplitudes – assume no significant interaction between fibre viscous boundary layers.
- Incremental scaling of thermal boundary layers amplitudes.



ROYAL INSTITUTE
OF TECHNOLOGY

Conclusions

- Poroelastic fibrous microstructure material model with analytical viscous drag force and thermal heat transfer expressions.
- Fibre diameter/orientation distributions, which are closely tied to material manufacturing processes.
- Acoustics vs temperature investigations.
- Traditional poroelastic transport properties are not required.
- COMSOL thermoviscous acoustic fluid simulations used as a *Virtual Laboratory* to confirm negligible interaction between the dominant viscous boundary layers.