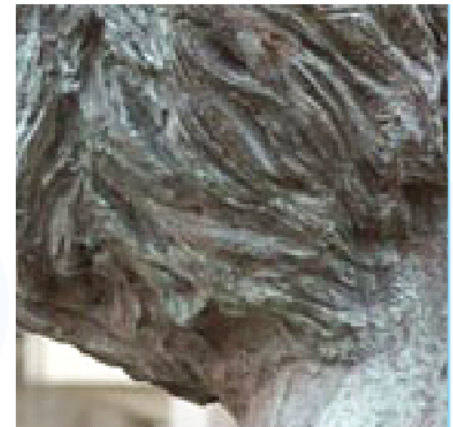
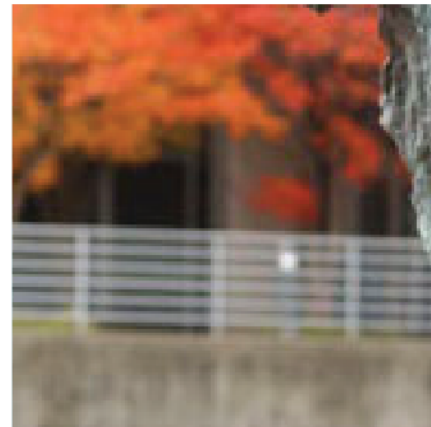




**OLD DOMINION**  
UNIVERSITY

IDEA FUSION



## Reynolds Number and Geometry Configuration Effect on Secondary Flows in S-Shaped Circular Bends

O. Ayala, M.F. Degenring, P. Loures

**COMSOL  
CONFERENCE**

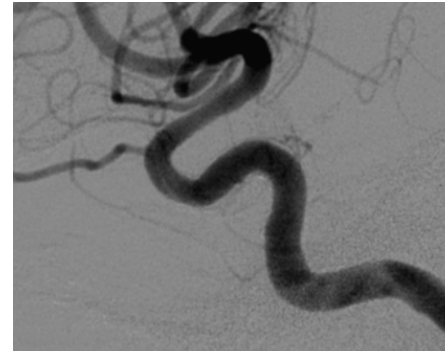
2016 BOSTON

October 6, 2016

CFD Session

# Introduction

S-shaped bends can be found in industrial pipe layouts and even in arteries!



- Presence of secondary flows
- If particles → erosion might be found



## ■ Most studies on laminar flows

(Hoogstraten et al., 1996; Johnston and Johnston, 2008; Niazmand and Jaghargh, 2010)

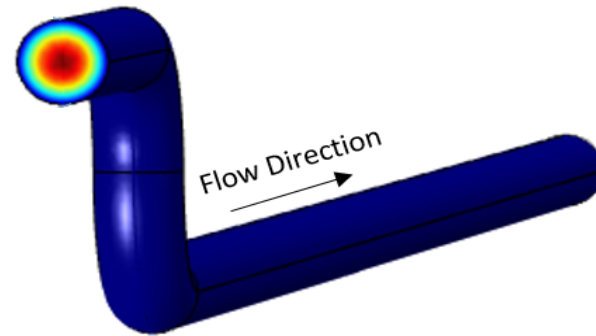
## ■ Few for turbulent flow but:

- non-circular cross sections (Ng and Birk, 2013; Debnath et al. 2015)
- primary flow interest only (Mazhar et al., 2014)
- limited Reynolds numbers (Taylor, 1984)

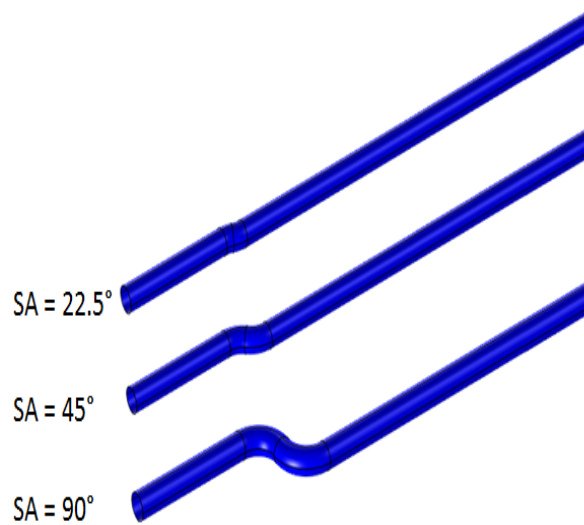
# Physical Model



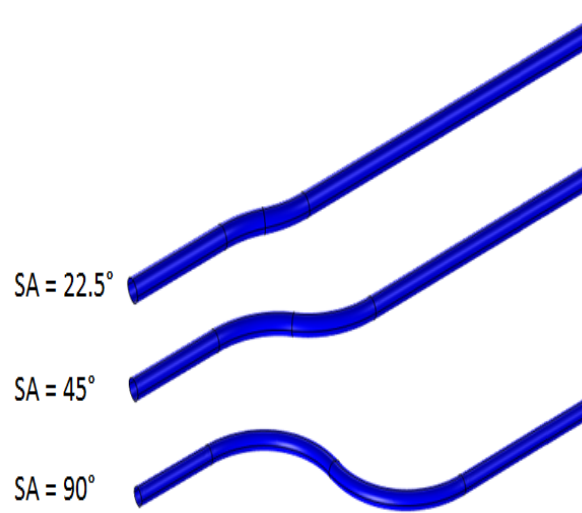
- Sweep Angle (SA)
- Radius of Curvature (RC)
- Reynolds Number



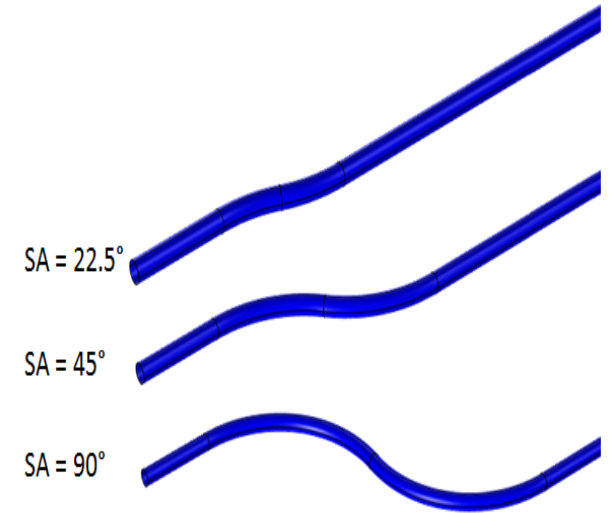
100, 1000, 10 000, 100 000



RC = 1.5D

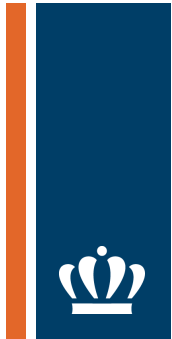


RC = 6.5D



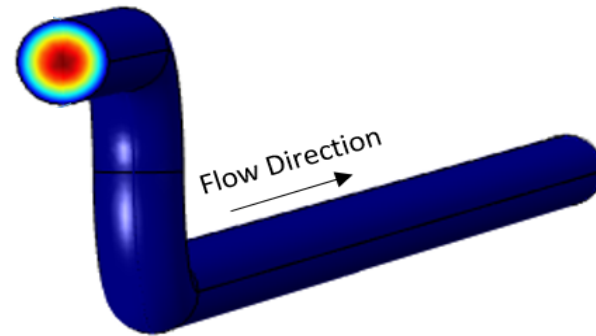
RC = 10D

# Mathematical and Numerical Model



$$\rho \frac{\partial \mathbf{v}}{\partial t} + \rho \mathbf{v} \cdot \nabla \mathbf{v} = \mu \nabla^2 \mathbf{v} - \nabla P$$
$$\nabla \cdot \mathbf{v} = 0$$

$k$ - $\epsilon$  model



■ INLET

Velocity profile

COMSOL

■ OUTLET

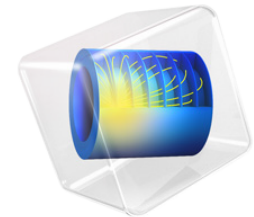
Constant pressure

■ WALL

Non-slip or Wall functions

■ STRAIGHT PIPES

Long enough (sensitivity)



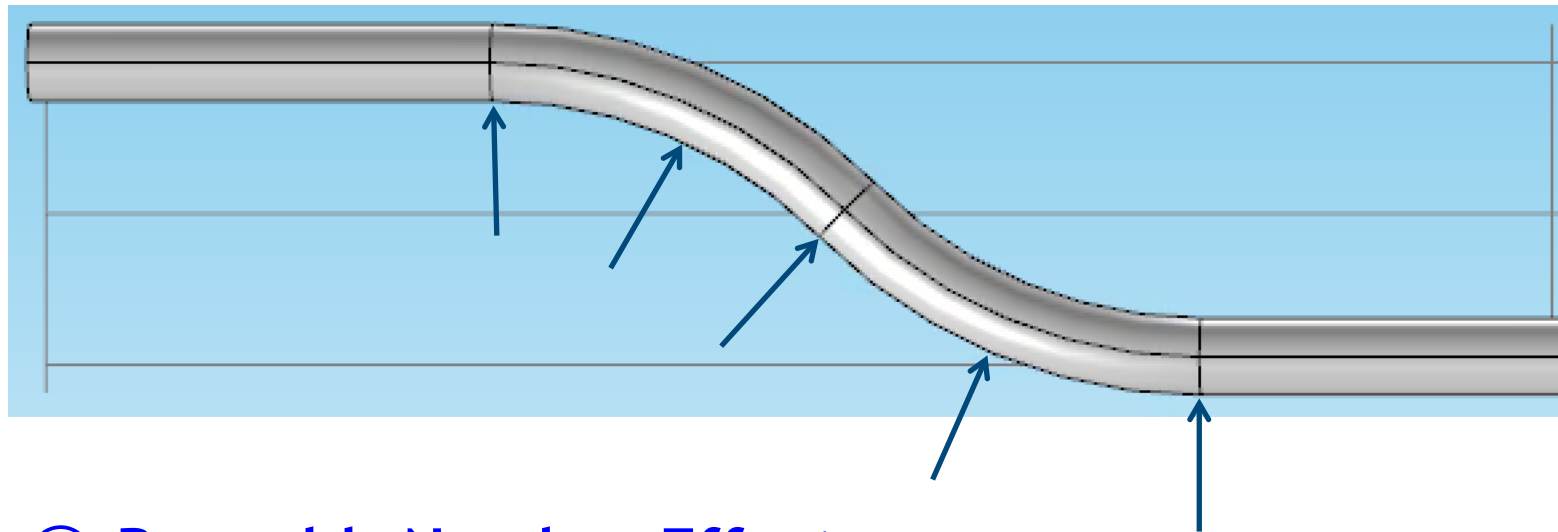
## MESH SENSITIVITY

(normal mesh  $\rightarrow$  5% diff,  $y^+ \sim 12$ )

## VALIDATION

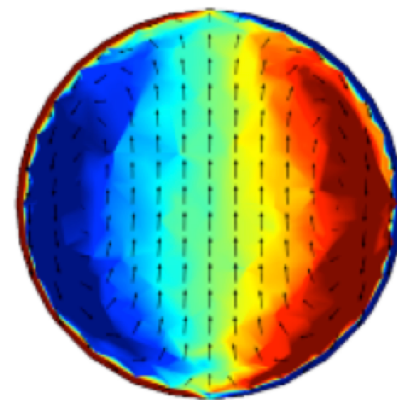
(Niazmand and Jaghargh, 2010)

# Results



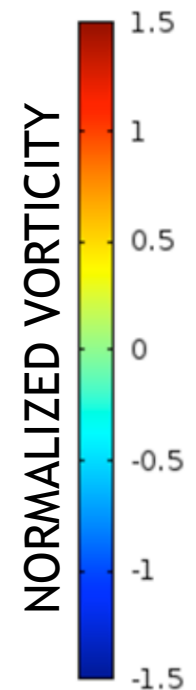
- ① Reynolds Number Effect
- ② Radius of Curvature Effect
- ③ Sweep Angle Effect

- a) Velocity vectors
- b) Normalized max velocity
- c) Normalized vorticity



31.7%

Normalized  
max velocity

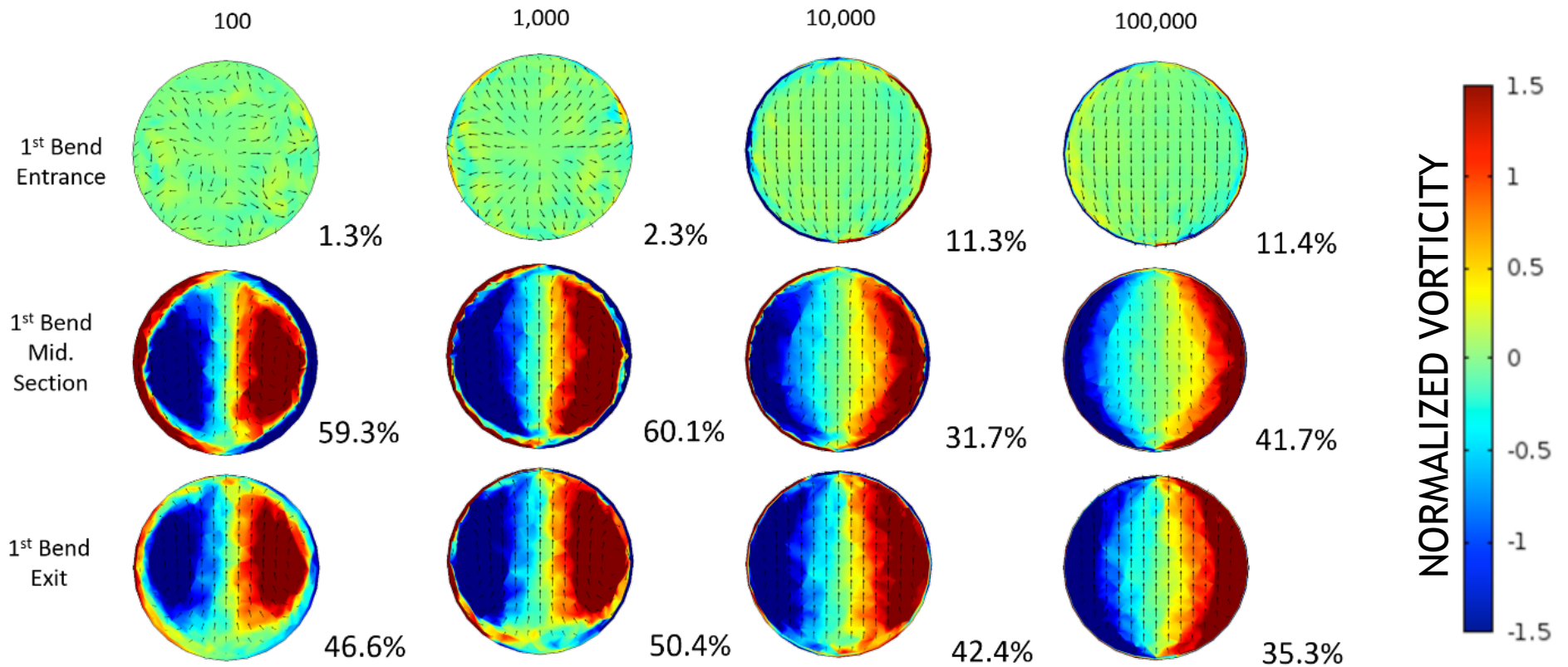


# Results

## Reynolds Number Effect



Curvature Ratio: 1.5D & Sweep Angle: 90°



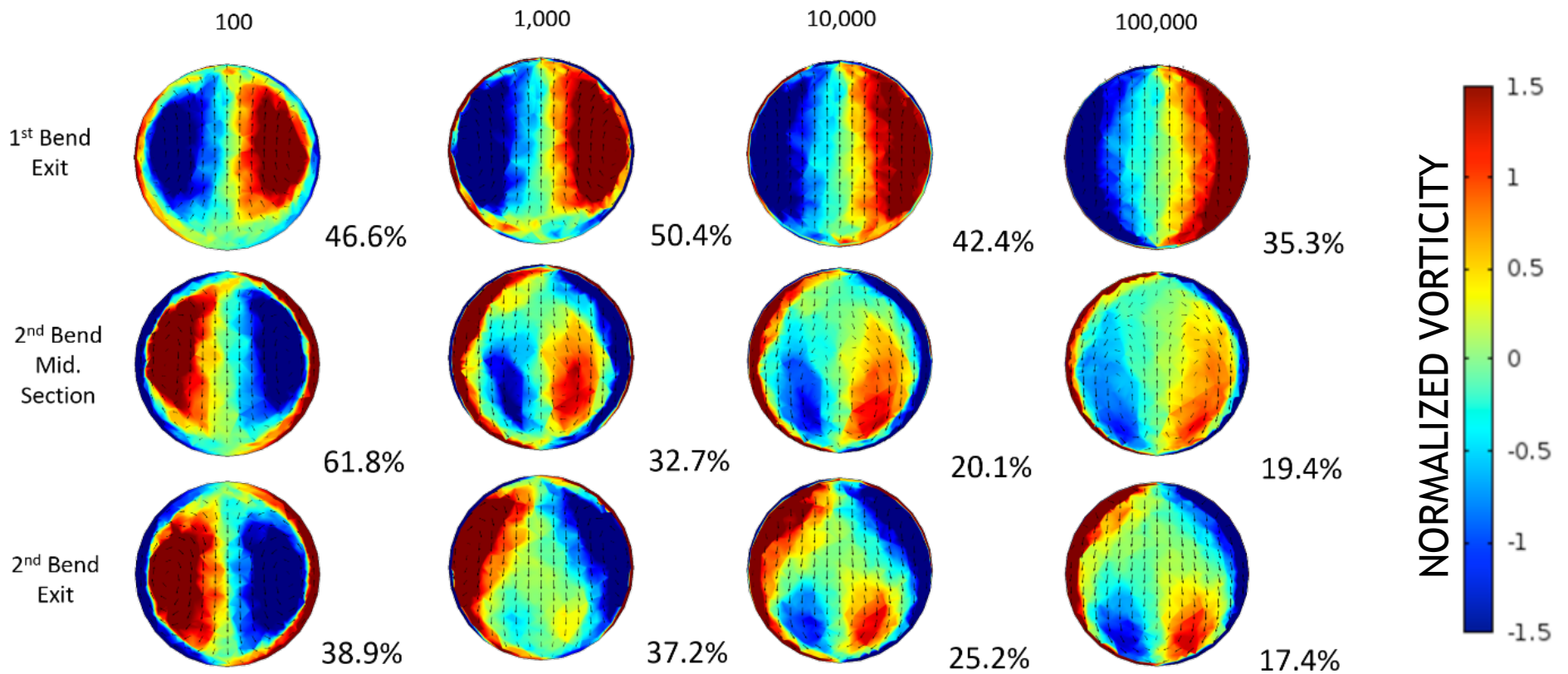
$$a_c \propto \left( \frac{u_{axial}^2}{r} \right)$$

Normalized max velocity

# Results Reynolds Number Effect



Curvature Ratio: 1.5D & Sweep Angle: 90°



$$a_c \propto \left( \frac{u_{axial}^2}{r} \right)$$

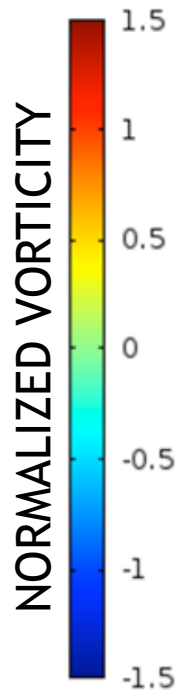
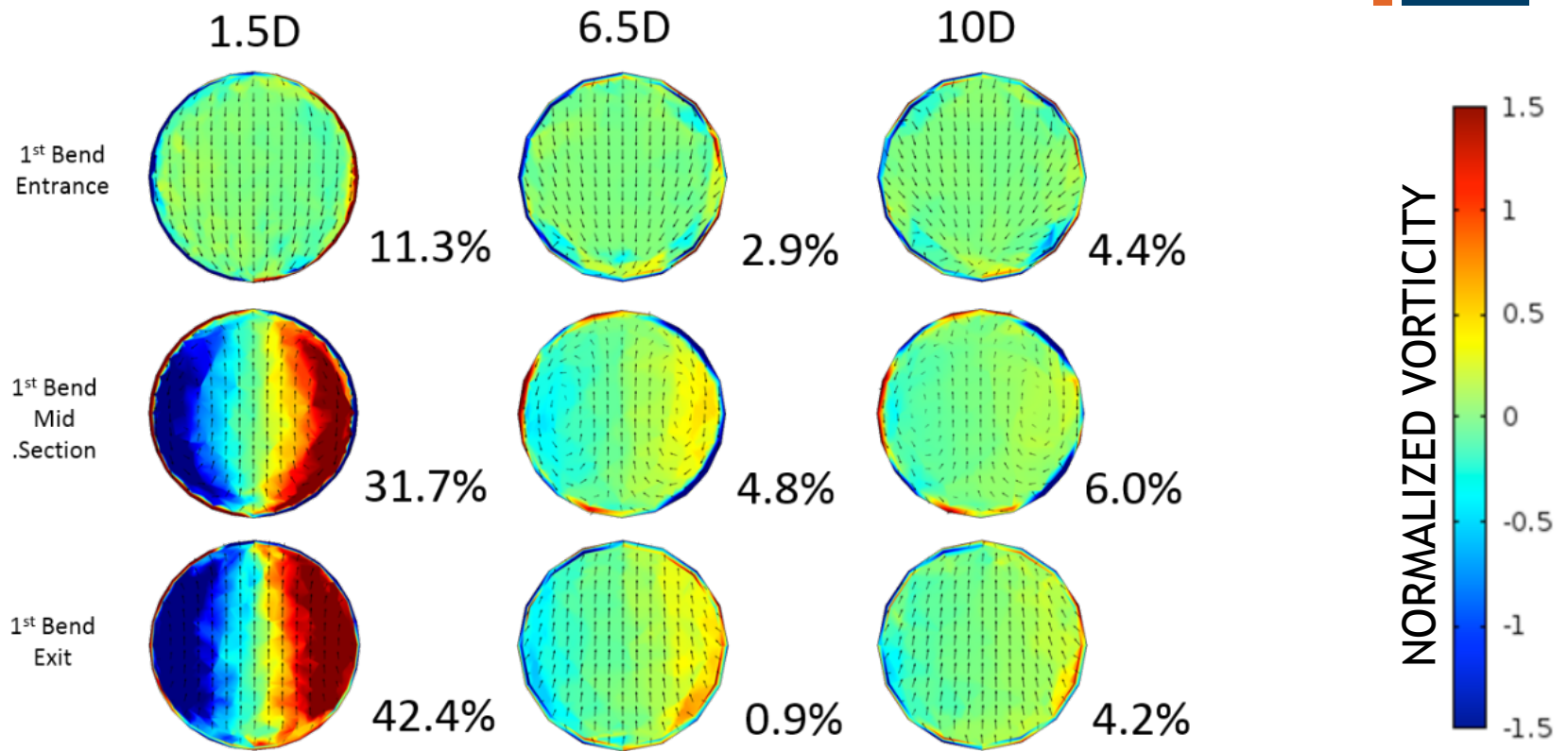
Normalized max velocity

# Results

## Radius of Curvature Effect



Reynolds Number: 10,000 & Sweep Angle: 90°



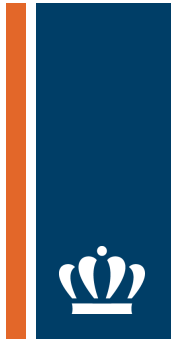
$$a_c \propto \left( \frac{u_{axial}^2}{r} \right)$$

Normalized max velocity

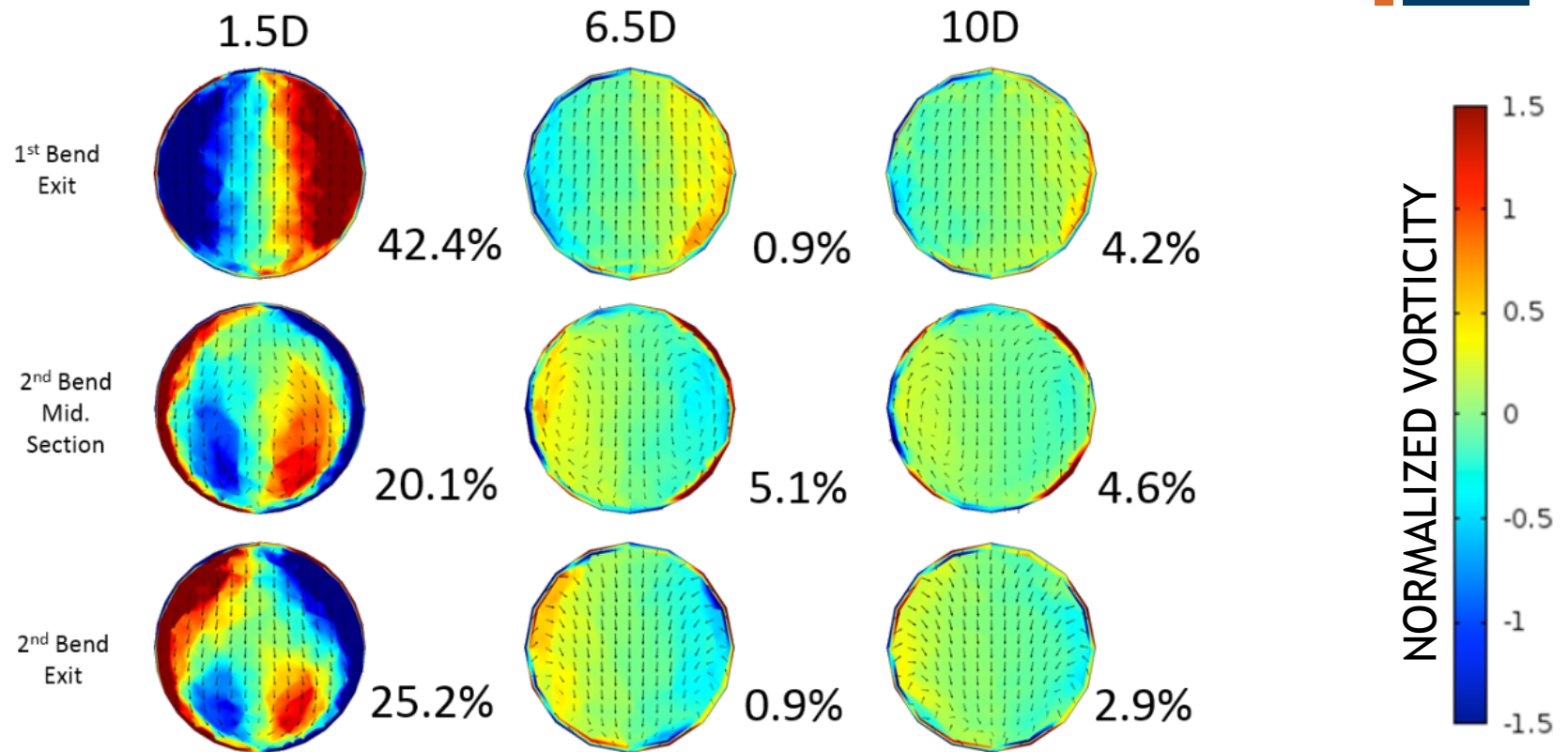


# Results

## Radius of Curvature Effect



Reynolds Number: 10,000 & Sweep Angle: 90°



$$a_c \propto \left( \frac{u_{axial}^2}{r} \right)$$

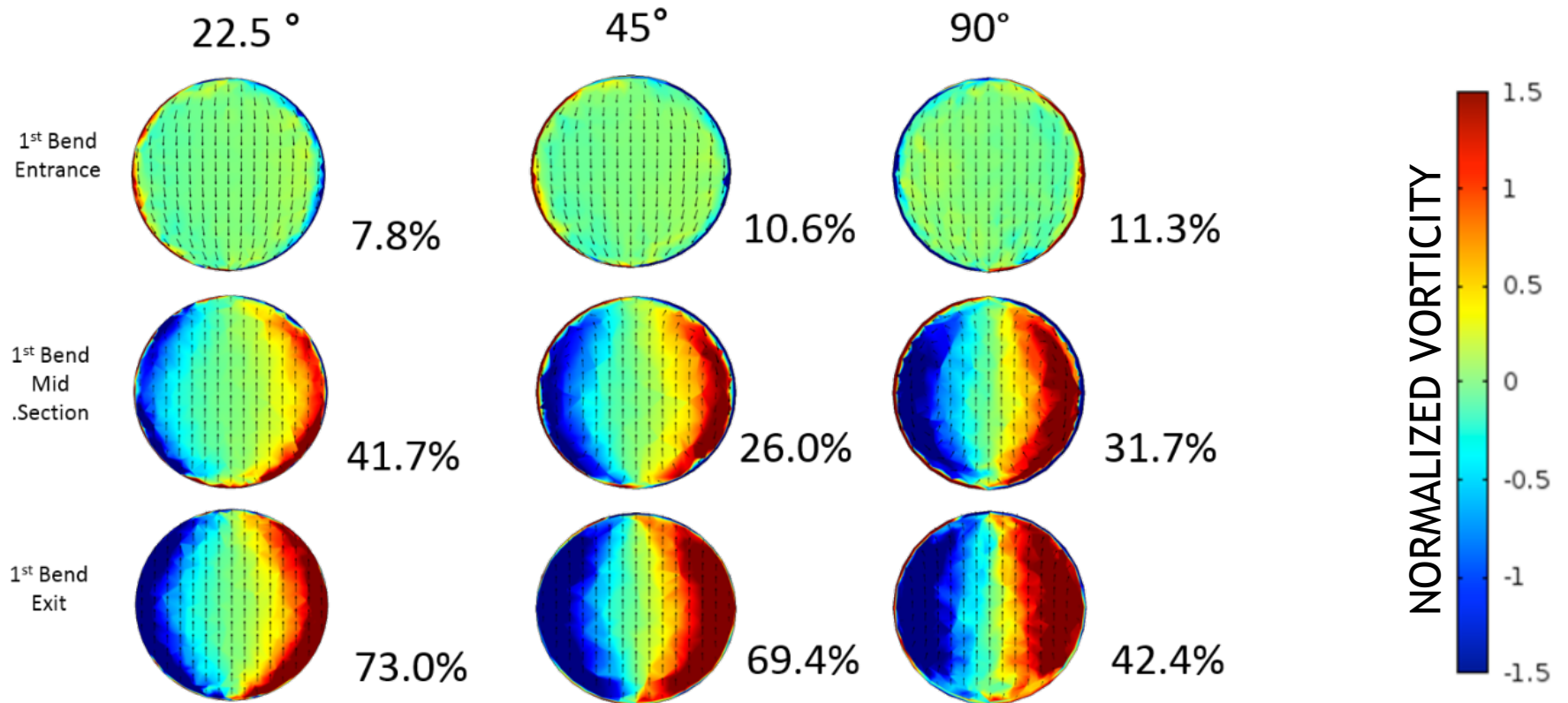
Normalized max velocity

# Results

## Sweep Angle Effect



Reynolds Number: 10,000 & Curvature Ratio: 1.5D

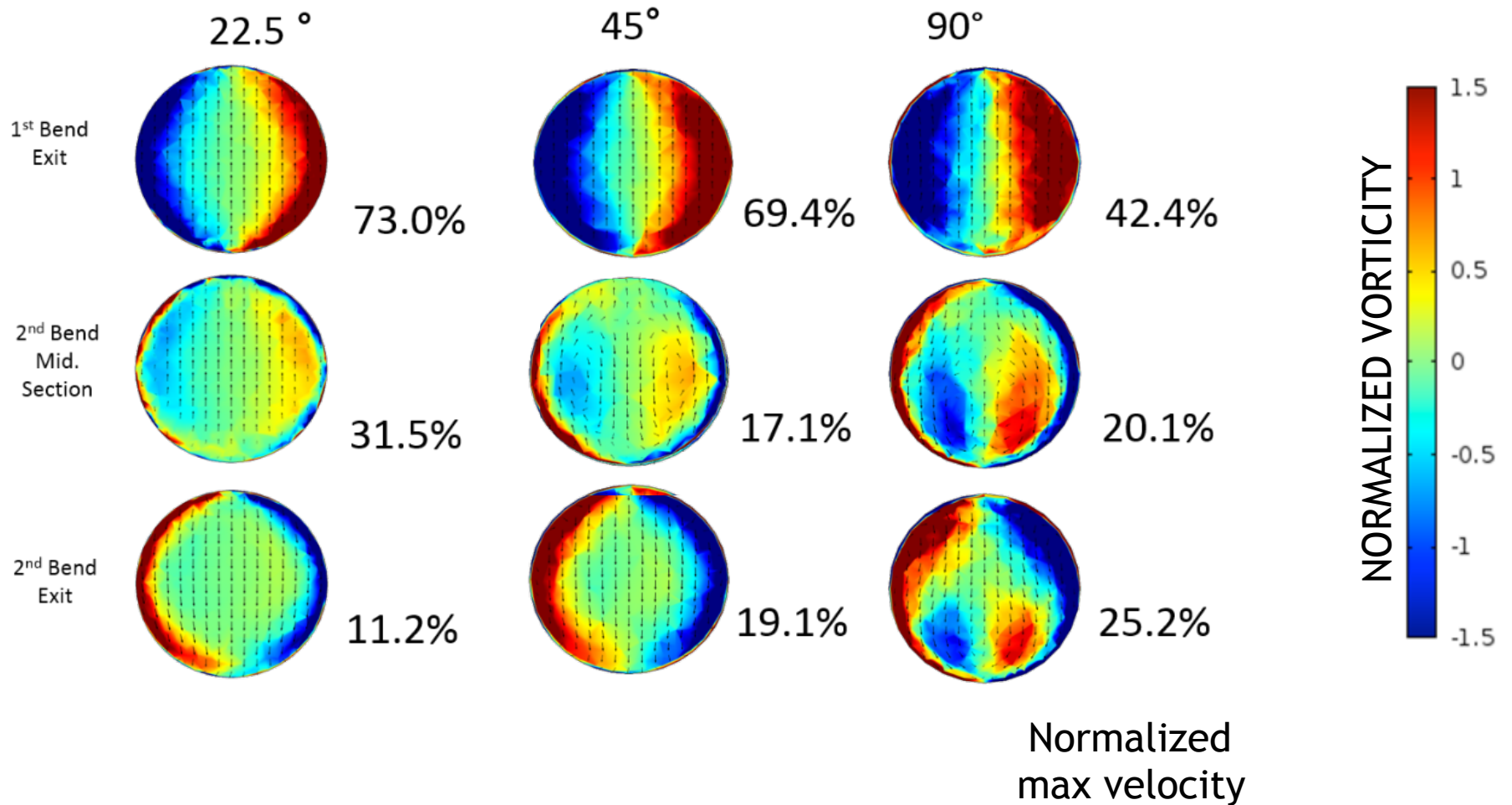


# Results

## Sweep Angle Effect



Reynolds Number: 10,000 & Curvature Ratio: 1.5D



# Conclusions



## ■ Reynolds Numbers

- 1<sup>st</sup> bend: High Re → 1<sup>st</sup> vorticity pair stays longer & closer to wall
- 2<sup>nd</sup> bend: High Re → secondary flow from 1<sup>st</sup> bend dominates

## ■ Radius of Curvature

- The smaller it is, the stronger the vorticity magnitude
- The smaller it is, the harder it takes for 2<sup>nd</sup> vortical pair to develop

## ■ Sweep Angle

- The larger it is, the stronger the vorticity magnitude is and the longer it lasts.

**OUR NEXT GOAL: Better quantification to develop models**