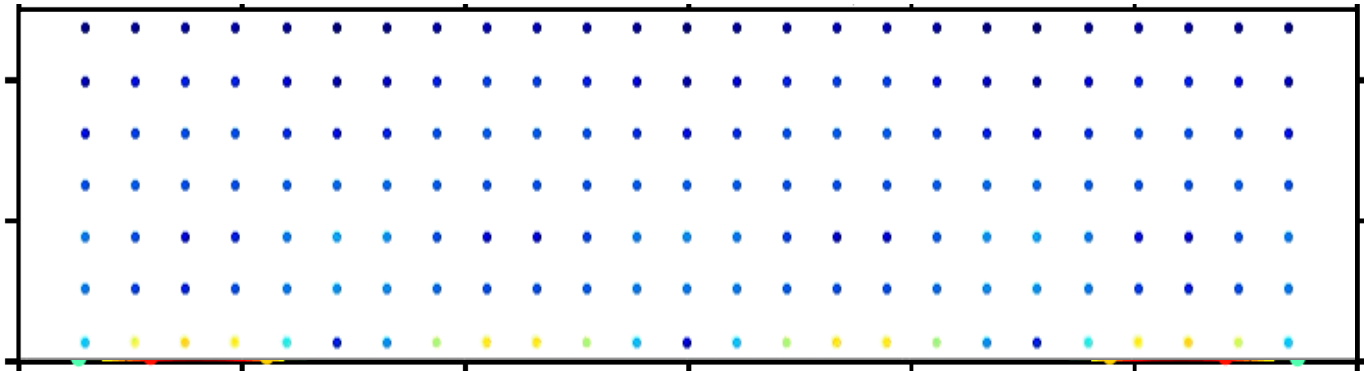


Simulation of SAW-Driven Microparticle Acoustophoresis Using COMSOL Multiphysics®



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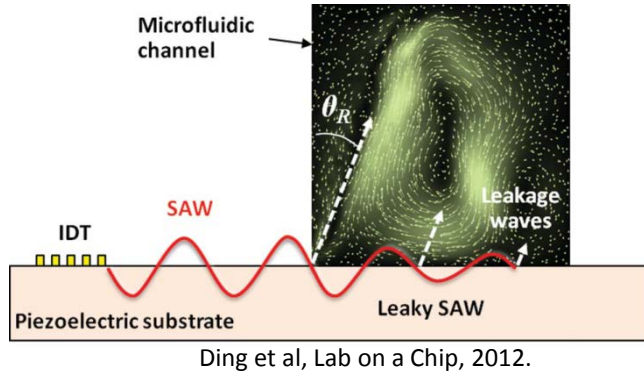
Outline

- **Introduction**
- **Numerical scheme**
- **Model validation**
- **COMSOL Modeling and convergence**
- **Results**
- **Conclusion and Outlook**

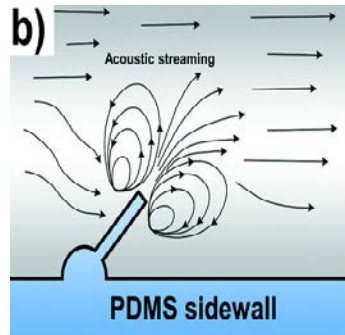
Acoustofluidics

Integration of acoustics with microfluidics

Acoustic Streaming



Tatsuno et al, Album Fluid Motion, 1982.

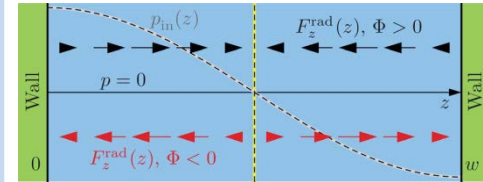
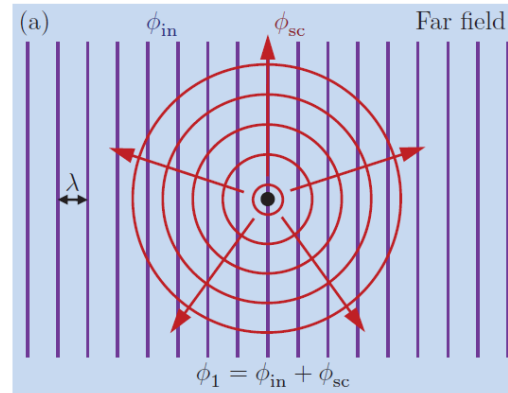


Huang et al, Lab on a Chip, 2014.

- Oscillating boundaries
- Time-averaged motion of the fluid

Useful for fluid and particle manipulation

Acoustic Radiation Force



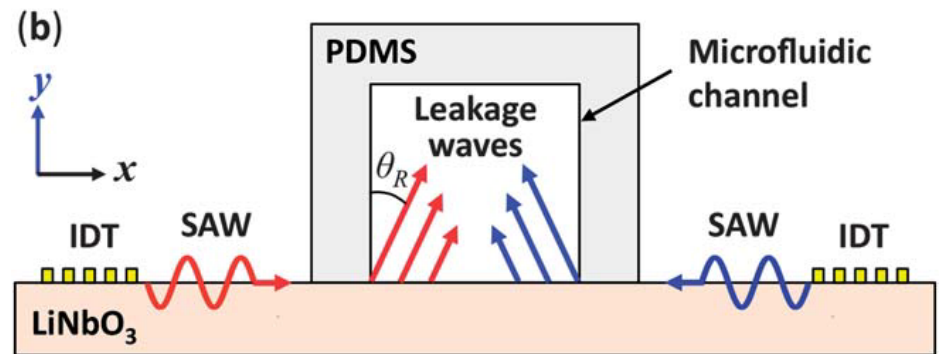
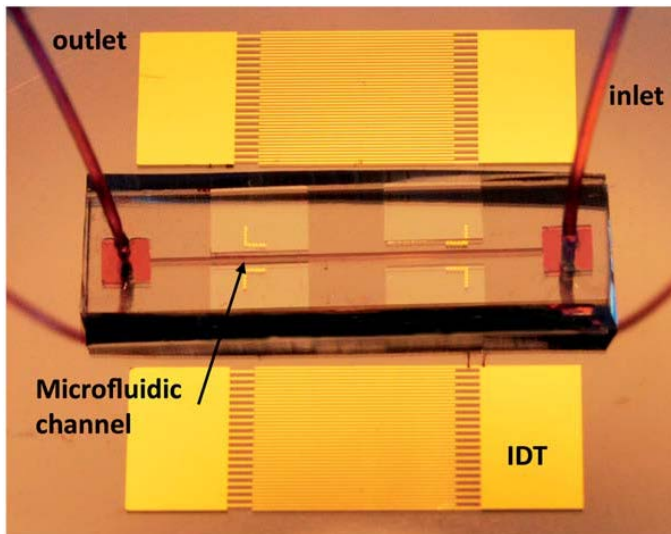
$$\mathbf{F}^{rad} = -\pi a^3 \left[\frac{2\kappa_0}{3} \text{Re}[f_1^* p_{in}^* \nabla p_{in}] - \rho_0 \text{Re}[f_2^* \mathbf{v}_{in}^* \cdot \nabla \mathbf{v}_{in}] \right]$$

M. Settnes and H. Bruus, Phys Rev E, 2012.

- Scattering of incident acoustic waves from the particles
- ARF acting towards pressure node or antinode

Surface Acoustic Wave (SAW)

- Surface Acoustic Wave (SAW) systems have gained prominence for various lab-on-a-chip applications.



Typical SAW device

Questions:

- Type of acoustic fields setup inside the channel?
- Effect of PDMS walls as opposed to harder materials (e.g. silicon)
- Critical transition size for particle motion inside the microchannel?

Governing equations

Balance of mass

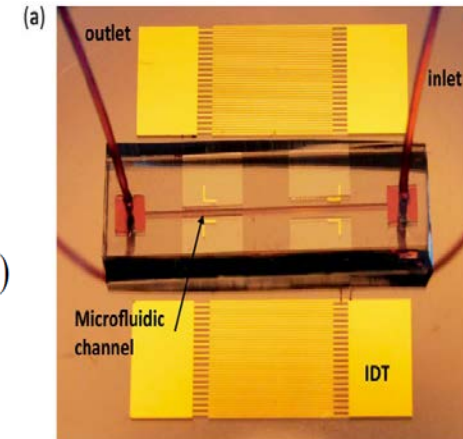
$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

Balance of linear momentum

$$\rho \frac{\partial \mathbf{v}}{\partial t} + \rho (\mathbf{v} \cdot \nabla) \mathbf{v} = -\nabla p + \mu \nabla^2 \mathbf{v} + \left(\mu_b + \frac{1}{3} \mu \right) \nabla (\nabla \cdot \mathbf{v})$$

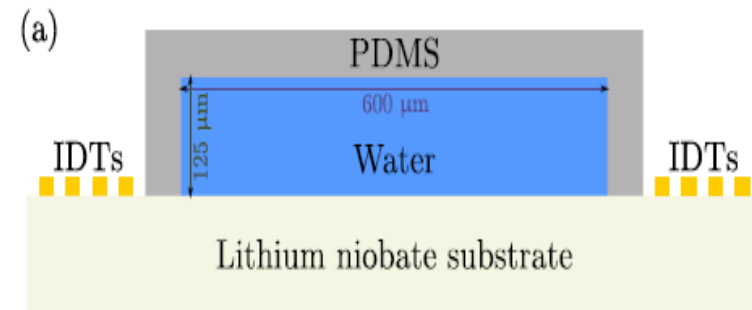
Constitutive relation

$$p = c_0^2 \rho$$



Numerical Challenges:

- Widely separated time scales – Characteristic oscillation period (10^{-7} s) vs. characteristic times dictated by streaming speeds (10^{-1} s)



Numerical Model

Perturbation expansion

$$\begin{aligned} \mathbf{v} &= \mathbf{v}_0 + \varepsilon \tilde{\mathbf{v}}_1 + \varepsilon^2 \tilde{\mathbf{v}}_2 + O(\varepsilon^3) + \dots \\ p &= p_0 + \varepsilon \tilde{p}_1 + \varepsilon^2 \tilde{p}_2 + O(\varepsilon^3) + \dots \\ \rho &= \rho_0 + \varepsilon \tilde{\rho}_1 + \varepsilon^2 \tilde{\rho}_2 + O(\varepsilon^3) + \dots \end{aligned}$$

First-order equations

$$\begin{aligned} \frac{\partial \rho_1}{\partial t} + \rho_0 (\nabla \cdot \mathbf{v}_1) &= 0, \\ \rho_0 \frac{\partial \mathbf{v}_1}{\partial t} &= -\nabla p_1 + \mu \nabla^2 \mathbf{v}_1 + \left(\mu_b + \frac{1}{3}\mu\right) \nabla (\nabla \cdot \mathbf{v}_1) \end{aligned}$$

Isentropic,
compressible
Newtonian fluid

No background
flow

$$\left\langle \frac{\partial \rho_2}{\partial t} \right\rangle + \rho_0 \nabla \cdot \langle \mathbf{v}_2 \rangle = -\nabla \cdot \langle \rho_1 \mathbf{v}_1 \rangle$$

Second-order equations

$$\rho_0 \left\langle \frac{\partial \mathbf{v}_2}{\partial t} \right\rangle + \left\langle \rho_1 \frac{\partial \mathbf{v}_1}{\partial t} \right\rangle + \rho_0 \langle \mathbf{v}_1 \cdot \nabla \mathbf{v}_1 \rangle = -\nabla \langle p_2 \rangle + \mu \nabla^2 \langle \mathbf{v}_2 \rangle + \left(\mu_b + \frac{1}{3}\mu\right) \nabla \nabla \cdot \langle \mathbf{v}_2 \rangle$$

- The nonlinear problem is divided into two sets to linear equations which can be solved successively.
- Numerical Scheme: COMSOL Multiphysics 5.1 with P2-P1 elements for velocity and pressure.

Model System and Boundary conditions

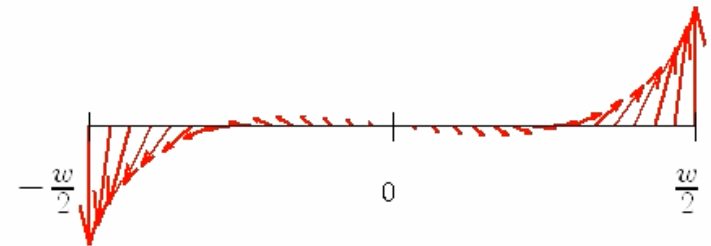
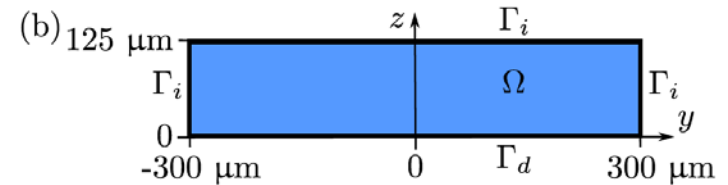
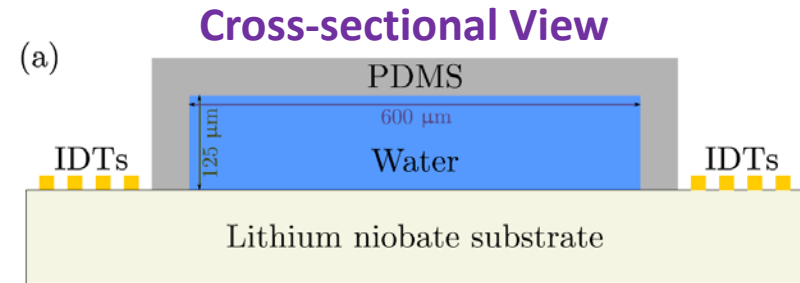
Impedance condition on the channel walls (PDMS)

$$\mathbf{n} \cdot \nabla p_1 = i \frac{\omega \rho_0}{\rho_m c_m} p_1$$

Standing SAW displacement on the bottom wall

$$u_y(t, y) = 0.6u_0 e^{-C_{d1}y} \left[\sin \left(\frac{-2\pi(y - w/2)}{\lambda} + \omega t - \Delta\phi \right) + \sin \left(\frac{-2\pi(w/2 - y)}{\lambda} + \omega t \right) \right],$$

$$u_z(t, y) = -u_0 e^{-C_{d1}y} \left[\cos \left(\frac{-2\pi(y - w/2)}{\lambda} + \omega t - \Delta\phi \right) + \cos \left(\frac{-2\pi(w/2 - y)}{\lambda} + \omega t \right) \right],$$



The displacement on the bottom wall was adopted from the results of numerical simulations by Gantner.

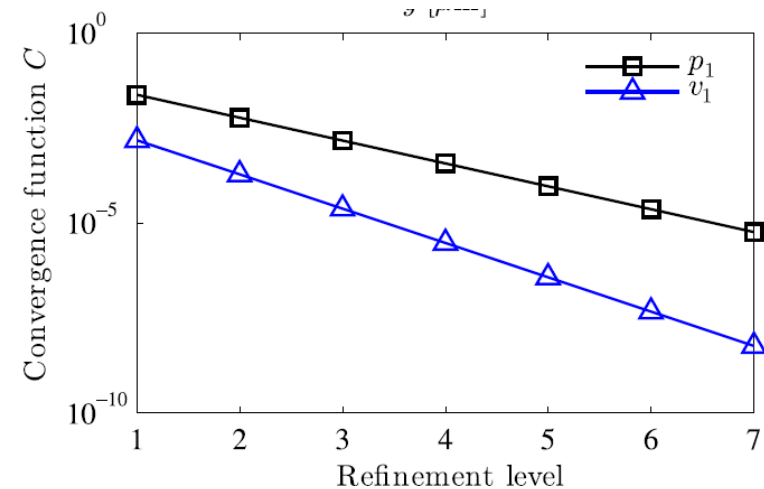
Model Validation

Method of Manufactured Solution

Step 1: Assume a solution for both pressure and velocity.

Step 2: Plug into PDEs to get the forcing and boundary conditions.

$$\begin{aligned}\mu_v \frac{\partial v}{\partial t} - \nu_1 \Delta v - \nu_2 \nabla(\nabla \cdot v) + \nabla p &= f_v && \text{in } \Omega, t > 0 \\ \mu_p \frac{\partial p}{\partial t} + \nabla \cdot v &= f_p && \text{in } \Omega, t > 0 \\ v &= g && \text{on } \partial\Omega, t > 0\end{aligned}$$



Step 3: Use the computed forcing and boundary conditions in the numerical model to obtain a solution.

Step 4: Compare the obtained solution with the assumed solution.

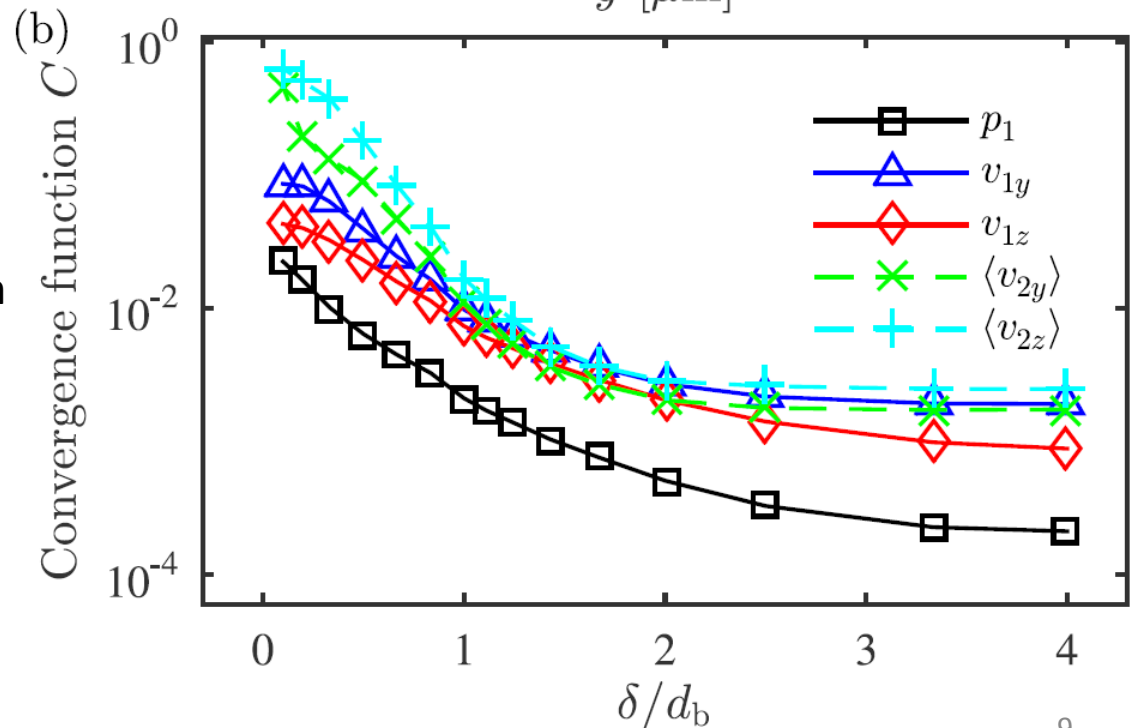
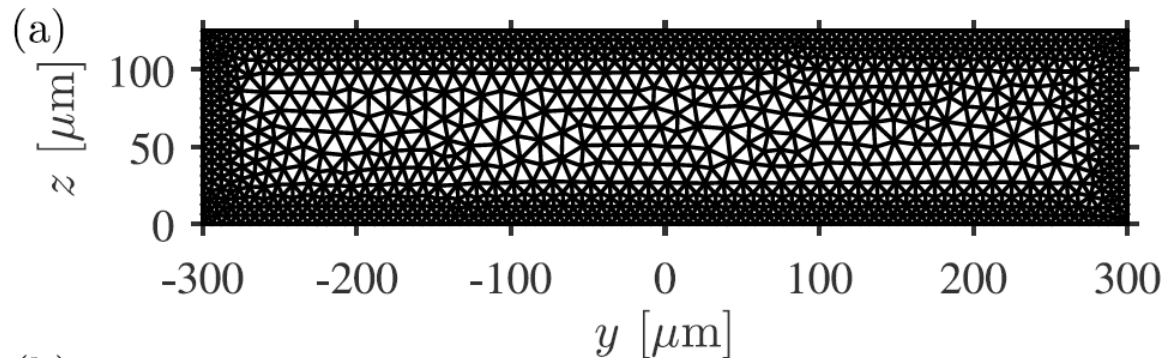
COMSOL Modeling and convergence

- *Weak PDE* interface
- Parametric sweep over mesh size.
- Finer mesh near the boundaries to resolve the boundary layers.

$$C(g) = \sqrt{\frac{\int (g - g_{\text{ref}})^2 dy dz}{\int (g_{\text{ref}})^2 dy dz}}$$

Here, g_{ref} = most refined solution

- Slower convergence of second-order fields since they depend on the gradients of first-order fields.

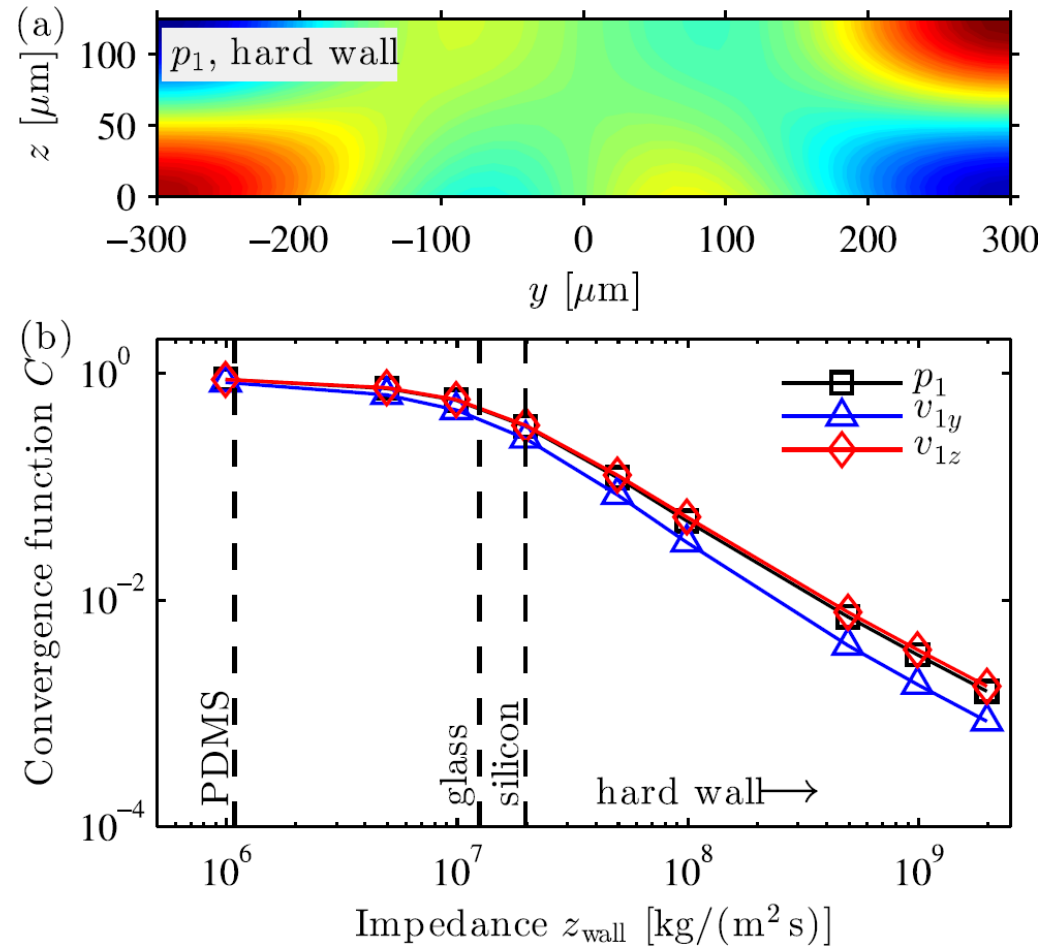


Impedance Sweep

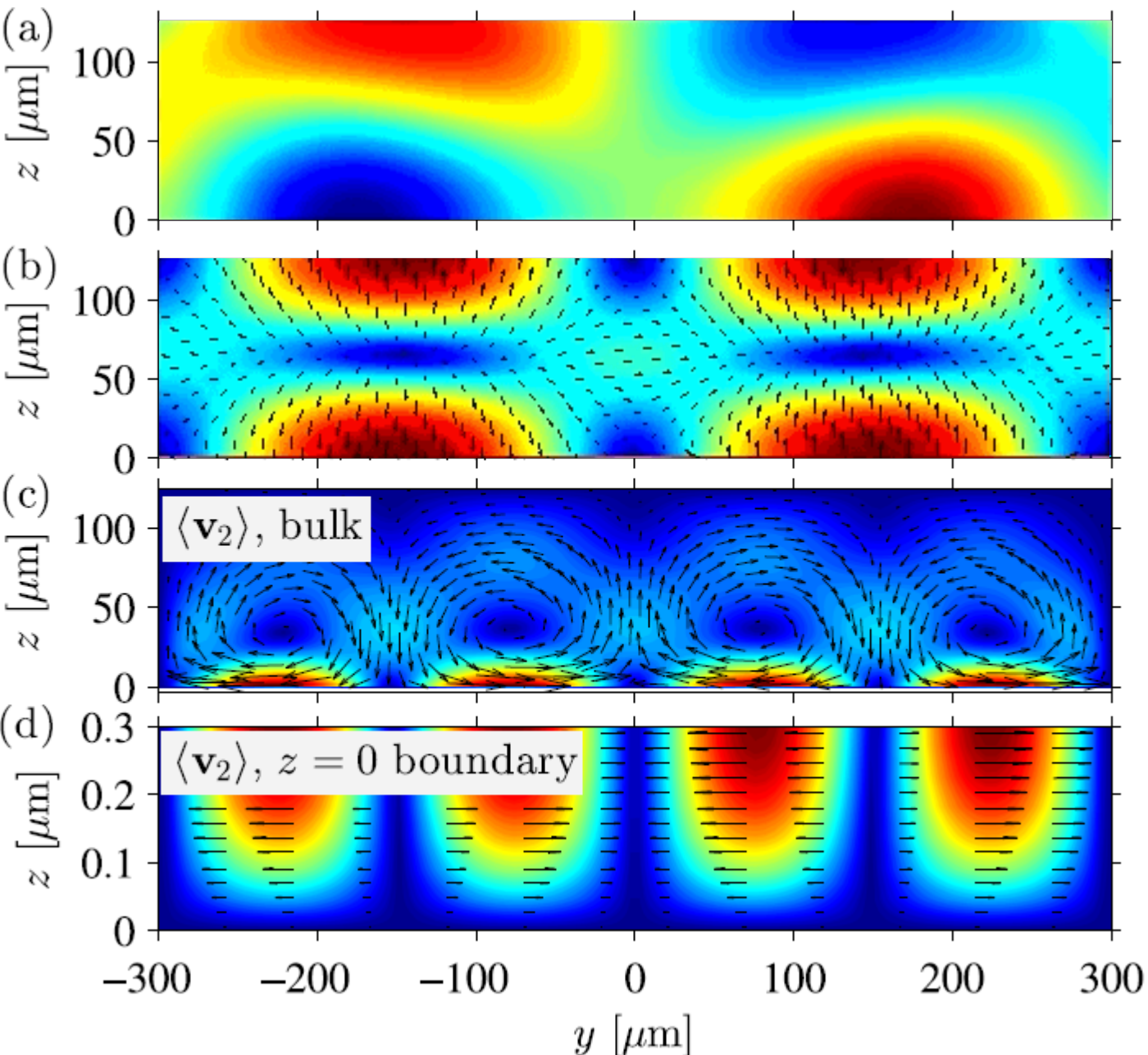
- Convergence to hard wall solution on increasing impedance of the wall.
- Resonances similar to hard wall system were observed for high values of impedances.
- Hard wall conditions suitable for bulk acoustic wave (BAW) systems but not SAW systems.

$$C(g) = \sqrt{\frac{\int (g - g_{\text{ref}})^2 dy dz}{\int (g_{\text{ref}})^2 dy dz}}$$

Here, g_{ref} = hard wall solution

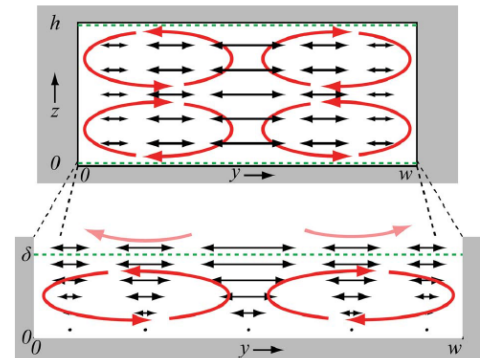


Acoustic fields



$$\mathbf{n} \cdot \nabla p_1 = i \frac{\omega \rho_0}{\rho_m c_m} p_1$$

Bulk Acoustic Wave (BAW) Device



P. Augustsson *et al*, Lab Chip, 2011, 11, 4152–4164.

$$\delta = \sqrt{\frac{2\nu}{\omega}}$$

Particle tracking

Radiation Force

$$\mathbf{F}^{\text{rad}} = -\pi a^3 \left[\frac{2\kappa_0}{3} \text{Re}[f_1^* p_1^* \nabla p_1] - \rho_0 \text{Re}[f_2^* \mathbf{v}_1^* \cdot \nabla \mathbf{v}_1] \right]$$

M. Settles and H. Bruus, Phys Rev E, vol. 85, p. 016327, 2012.

$$f_1 = 1 - \frac{\kappa_p}{\kappa_0} \quad \text{and} \quad f_2 = \frac{2(1 - \gamma)(\rho_p - \rho_0)}{2\rho_p + \rho_0(1 - 3\gamma)}$$

$$\gamma = -\frac{3}{2}[1 + \nu(1 + \tilde{\delta})]\tilde{\delta}, \quad \tilde{\delta} = \frac{\delta}{a}, \quad \delta = \sqrt{\frac{\mu}{\pi f \rho_0}}$$

Hydrodynamic Drag Force

$$\mathbf{F}^{\text{drag}} = 6\pi\mu a (\langle \mathbf{v}_2 \rangle - \mathbf{v}^{\text{bead}})$$

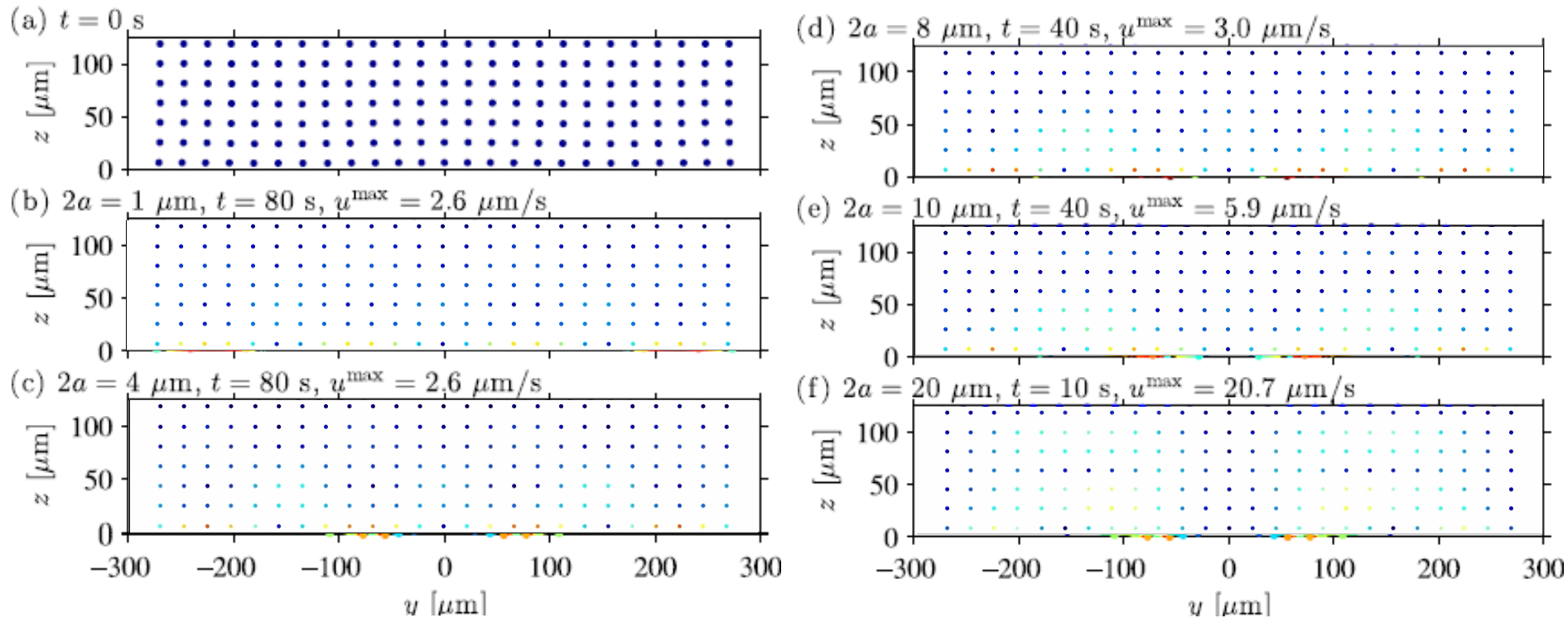
Newton's Second Law

$$m_p \mathbf{a}_p = \mathbf{F}^{\text{rad}} + \mathbf{F}^{\text{drag}}$$

Particle Velocity

$$\mathbf{v}^{\text{bead}} = \langle \mathbf{v}_2 \rangle + \frac{\mathbf{F}^{\text{rad}}}{6\pi\mu a}$$

Particle tracking

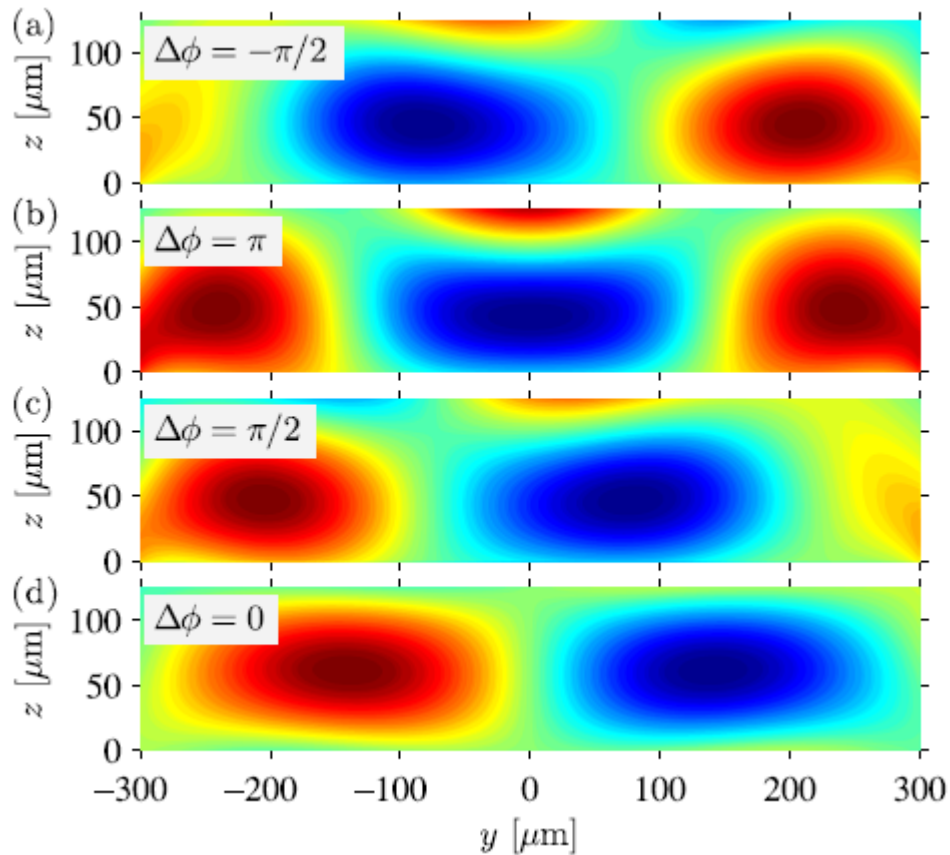


Side View

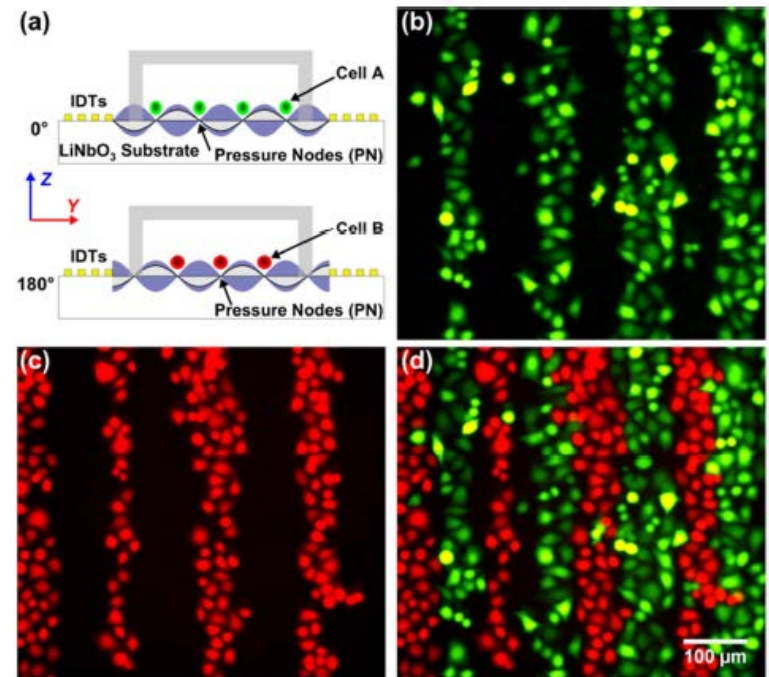


Vertical focusing:
probably due to gravity

Application: Phase Sweep



$$\Delta x = \frac{1}{2k} \varphi = \frac{\lambda}{720^\circ} \varphi \quad \varphi \in [0^\circ, 360^\circ]$$



Li *et al*, Lab Chip, Anal. Chem., 86, 9853–9859, 2014.

- Displacement of pressure node by changing the phase of one IDT

Conclusion and Outlook

- A numerical model for standing SAW based systems is presented.
- The findings are very different from the BAW systems
 - Traveling wave setup in the channel
 - Different boundary layer
 - Leakage of energy to PDMS
- The boundary layer phenomena is yet to be fully understood.
- Quantitative 3D APTV measurements for the experimental verification are in progress.
- More details about the model in the following article:

Nitesh Nama, Rune Barnkob, Zhangming Mao, Christian J. Kähler, Francesco Costanzo, and Tony Jun Huang, **Numerical study of acoustophoretic motion of particles in a PDMS microchannel driven by surface acoustic waves**, *Lab on a Chip*, Vol. 15, pp. 2700-2709, 2015.

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