## Non-Isothermal MEMS Heat Exchanger

## Introduction

The following example builds and solves a conduction and convection heat transfer problem using the Non-Isothermal Flow physics.

The example concerns a stainless-steel MEMS heat exchanger, which you can find in lab-on-a-chip devices in biotechnology and in microreactors such as for micro-fuel cells. This model examines the heat exchanger in 3D, and it involves heat transfer through both convection and conduction.

## Model Definition-Heat Exchanger

Figure 1 shows the geometry of the heat exchanger. It is necessary to model only one unit cell because they are all almost identical except for edge effects in the outer cells.


Figure 1: Depiction of the modeled part of the heat exchanger (left).
The governing equation for this model is the heat equation for conductive and convective heat transfer

$$
\begin{equation*}
\nabla \cdot(-k \nabla T)=Q-\rho C_{p} \mathbf{u} \cdot \nabla T \tag{1}
\end{equation*}
$$

where $C_{p}$ denotes the specific heat capacity ( $\left.\mathrm{J} /(\mathrm{kg} \cdot \mathrm{K})\right), T$ is the temperature $(\mathrm{K}), k$ is the thermal conductivity $(\mathrm{W} /(\mathrm{m} \cdot \mathrm{K})), \rho$ is the density $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$, $\mathbf{u}$ is the velocity vector $(\mathrm{m} / \mathrm{s})$, and $Q$ is a sink or source term (which you set to zero because there is no production or consumption of heat in the device).

In the solid part of the heat exchanger the velocity vector, $\mathbf{u}=(u, v, w)$, is set to zero in all directions. In the channels the velocity field is defined by an analytical expression that approximates fully-developed laminar flow for a circular cross section. For both the hot and cold streams, you set the velocity components in the $x$ and $z$ directions to zero.

For the hot stream, the expression

$$
\begin{equation*}
v=v_{\max }\left(1-\left(\frac{r}{R}\right)^{2}\right) \tag{2}
\end{equation*}
$$

gives the $y$-component of the velocity where

- $v_{\text {max }}$ is the maximum velocity $(\mathrm{m} / \mathrm{s})$, which arises in the middle of the channel
- $r$ is the distance from the center of the channel (m)
- $R$ is the channel radius ( m )

You describe velocity in the cold stream in the same manner but in the opposite direction

$$
\begin{equation*}
v=-v_{\max }\left(1-\left(\frac{r}{R}\right)^{2}\right) \tag{3}
\end{equation*}
$$

In an extended approach, instead of using the analytical expression for the velocity field, the fluid in the channels can be simulated using the Non-Isothermal Flow physics interface. Here the density is defined as

$$
\begin{equation*}
\rho=\rho_{\mathrm{m}}\left(1-\frac{T-T_{\mathrm{m}}}{T_{\mathrm{m}}}\right) \tag{4}
\end{equation*}
$$

where $\rho_{\mathrm{m}}$ the mean density $\left(\mathrm{kg} / \mathrm{m}^{3}\right), T$ is the temperature $(\mathrm{K})$, and $T_{\mathrm{m}}=\left(T_{\text {cold }}+T_{\text {hot }}\right) / 2$ is the mean fluid temperature.

The boundary conditions are insulating for all outer surfaces except for the inlet and outlet boundaries in the fluid channels. At the inlets, you specify constant temperatures for the cold and hot streams, $T_{\text {cold }}$ and $T_{\text {hot }}$, respectively. At the outlets, convection dominates the transport of heat so you apply the convective flux boundary condition:

$$
-k \nabla T \cdot \mathbf{n}=0
$$

## Results and Discussion

Figure 2 shows the temperature isosurfaces and the heat flux streamlines for the conductive heat flux in the device. The temperature isosurfaces clearly show the convective term's influence in the channels. Figure 3 displays the corresponding results for the extended model. As the plot shows, the temperature distribution is very similar to that in the first study, which can therefore be concluded to be a good approximation of the extended case.

Isosurface: Temperature (K) Streamline: Conductive heat flux


Figure 2: Isotherms and conductive heat flux streamlines in the cell unit's geometry.


Figure 3: Extended model results; isotherms and conductive heat flux streamlines in the cell unit's geometry.

```
Model Library path: Heat_Transfer_Module/Heat_Exchangers/
heat_exchanger_ni
```


## Modeling Instruction

From the File menu, choose New.

## NEW

I In the New window, click the Model Wizard button.

## MODEL WIZARD

I In the Model Wizard window, click the 3D button.
2 In the Select physics tree, select Fluid Flow $>$ Non-Isothermal Flow>Laminar Flow (nitf).
3 Click the Add button.

4 Click the Study button.
5 In the tree, select Preset Studies>Stationary.
6 Click the Done button.

## GLOBAL DEFINITIONS

## Parameters

I On the Home toolbar, click Parameters.
2 In the Parameters settings window, locate the Parameters section.
3 In the table, enter the following settings:

| Name | Expression | Value | Description |
| :--- | :--- | :--- | :--- |
| R | $50[\mathrm{um}]$ | $5.000 \mathrm{E}-5 \mathrm{~m}$ | Channel radius |
| v_mean | $2.5[\mathrm{~mm} / \mathrm{s}]$ | $0.002500 \mathrm{~m} / \mathrm{s}$ | Mean velocity |
| T_hot | $330[\mathrm{~K}]$ | 330.0 K | Temperature, hot <br> channel |
| T_cold | $300[\mathrm{~K}]$ | 300.0 K | Temperature, cold <br> channel |
| T_mean | $($ T_hot+T_cold)/2 | 315.0 K | Mean temperature |
| rho_mean_w | $1000\left[\mathrm{~kg} / \mathrm{m}^{\wedge} 3\right]$ | $1000 \mathrm{~kg} / \mathrm{m}^{3}$ | Fluid mean density |

## GEOMETRY I

I In the Model Builder window, under Component I click Geometry I.
2 In the Geometry settings window, locate the Units section.
3 From the Length unit list, choose $\boldsymbol{\mu m}$.

## Work Plane I

I On the Geometry toolbar, click Work Plane.
2 In the Work Plane settings window, locate the Plane Definition section.
3 From the Plane list, choose zx-plane.

## Rectangle I

I In the Model Builder window, under Component I>Geometry I>Work Plane I right-click Plane Geometry and choose Rectangle.

2 In the Rectangle settings window, locate the Size section.
3 In the Width edit field, type 300.
4 In the Height edit field, type 100.

5 Click the Build Selected button.
6 Click the Zoom Extents button on the Graphics toolbar.
Circle 1
I Right-click Plane Geometry and choose Circle.
2 In the Circle settings window, locate the Size and Shape section.
3 In the Radius edit field, type R.
4 In the Sector angle edit field, type 180.
5 Locate the Position section. In the xw edit field, type 100.
6 Click the Build Selected button.

## Circle 2

I Right-click Plane Geometry and choose Circle.
2 In the Circle settings window, locate the Size and Shape section.
3 In the Radius edit field, type R.
4 In the Sector angle edit field, type 180.
5 Locate the Position section. In the xw edit field, type 200.
6 In the yw edit field, type 100.
7 Locate the Rotation Angle section. In the Rotation edit field, type 180.
8 Click the Build Selected button.
9 On the Work Plane toolbar, click Close.

## Extrude I

I On the Geometry toolbar, click Extrude.
2 In the Extrude settings window, locate the Distances from Plane section.
3 In the table, enter the following settings:

## Distances ( $\mu \mathrm{m}$ )

400
4 Click the Build Selected button.
5 Click the Zoom Extents button on the Graphics toolbar.
Ignore Edges I
I On the Geometry toolbar, click Virtual Operations and choose Ignore Edges.
2 On the object fin, select Edges 8 and 33 only.

3 Click the Build Selected button.


## DEFINITIONS

## Explicit I

I On the Definitions toolbar, click Explicit.
2 In the Model Builder window, under Component I>Definitions right-click Explicit I and choose Rename.

3 Go to the Rename Explicit dialog box and type solid in the New name edit field.
4 Click OK.
5 Select Domain 1 only.

## Explicit 2

I On the Definitions toolbar, click Explicit.
2 Select Domains 2 and 3 only.
3 Right-click Component I>Definitions>Explicit 2 and choose Rename.
4 Go to the Rename Explicit dialog box and type channels in the New name edit field.
5 Click OK.

## Explicit 3

I On the Definitions toolbar, click Explicit.

2 Select Domain 2 only.
3 Right-click Component I>Definitions>Explicit 3 and choose Rename.
4 Go to the Rename Explicit dialog box and type hot channel in the New name edit field.

5 Click OK.

## Explicit 4

I On the Definitions toolbar, click Explicit.
2 Select Domain 3 only.
3 Right-click Component I>Definitions>Explicit 4 and choose Rename.
4 Go to the Rename Explicit dialog box and type cold channel in the New name edit field.

5 Click OK.

## Explicit 5

I On the Definitions toolbar, click Explicit.
2 In the Explicit settings window, locate the Input Entities section.
3 From the Geometric entity level list, choose Boundary.
4 Select Boundaries $6,8,12$, and 13 only.
5 Right-click Component I>Definitions>Explicit 5 and choose Rename.
6 Go to the Rename Explicit dialog box and type channel walls in the New name edit field.

## 7 Click OK.

Variables I
I Right-click Definitions and choose Variables.
2 In the Variables settings window, locate the Geometric Entity Selection section.
3 From the Geometric entity level list, choose Domain.
4 From the Selection list, choose channels.
5 Locate the Variables section. In the table, enter the following settings:

| Name | Expression | Unit | Description |
| :--- | :--- | :--- | :--- |
| rho_w | rho_mean_w* $\left(1-\left(T-T \_m e a n\right) / T \_m e a n\right)$ |  | Fluid density |

## MATERIALS

On the Home toolbar, click Add Material.

## ADD MATERIAL

I Go to the Add Material window.
2 In the tree, select Built-In>Steel AISI 4340.
3 In the Add material window, click Add to Component.

## MATERIALS

## Steel AISI 4340

I In the Model Builder window, under Component I>Materials click Steel AISI 4340.
2 In the Material settings window, locate the Geometric Entity Selection section.
3 From the Selection list, choose solid.

ADD MATERIAL
I Go to the Add Material window.
2 In the tree, select Built-In>Water, liquid.
3 In the Add material window, click Add to Component.

## MATERIALS

## Water, liquid

I In the Model Builder window, under Component I>Materials click Water, liquid.
2 In the Material settings window, locate the Geometric Entity Selection section.
3 From the Selection list, choose channels.

NON-ISOTHERMAL FLOW

## Fluid I

I In the Model Builder window, expand the Component I>Non-Isothermal Flow node, then click Fluid I.

2 In the Fluid settings window, locate the Thermodynamics section.
3 From the $\rho$ list, choose User defined. In the associated edit field, type rho_w.
4 From the $\gamma$ list, choose User defined. In the associated edit field, type 1.

## Heat Transfer in Solids I

I On the Physics toolbar, click Domains and choose Heat Transfer in Solids.
2 In the Heat Transfer in Solids settings window, locate the Domain Selection section.
3 From the Selection list, choose solid.

## Inlet I

I On the Physics toolbar, click Boundaries and choose Inlet.
2 Select Boundary 5 only.
3 In the Inlet settings window, locate the Boundary Condition section.
4 From the Boundary condition list, choose Laminar inflow.
5 Locate the Laminar Inflow section. In the $U_{\mathrm{av}}$ edit field, type $v$ _mean.

## Inlet 2

I On the Physics toolbar, click Boundaries and choose Inlet.
2 Select Boundary 15 only.
3 In the Inlet settings window, locate the Boundary Condition section.
4 From the Boundary condition list, choose Laminar inflow.
5 Locate the Laminar Inflow section. In the $U_{\text {av }}$ edit field, type v_mean.

## Outlet I

I On the Physics toolbar, click Boundaries and choose Outlet.
2 Select Boundaries 11 and 14 only.

## Symmetry, Flow I

I On the Physics toolbar, click Boundaries and choose Symmetry, Flow.
2 Select Boundaries 4 and 17 only.

## Temperature I

I On the Physics toolbar, click Boundaries and choose Temperature.
2 Select Boundary 5 only.
3 In the Temperature settings window, locate the Temperature section.
4 In the $T_{0}$ edit field, type T_hot.

## Temperature 2

I On the Physics toolbar, click Boundaries and choose Temperature.
2 Select Boundary 15 only.
3 In the Temperature settings window, locate the Temperature section.
4 In the $T_{0}$ edit field, type T_cold.
Outflow I
I On the Physics toolbar, click Boundaries and choose Outflow.
2 Select Boundaries 11 and 14 only.

## MESH I

## Free Triangular I

I In the Model Builder window, under Component I right-click Mesh I and choose More Operations>Free Triangular.

2 In the Free Triangular settings window, locate the Boundary Selection section.
3 From the Selection list, choose channel walls.
Size 1
I Right-click Component I>Mesh I>Free Triangular I and choose Size.
2 In the Size settings window, locate the Element Size section.
3 Click the Custom button.
4 Locate the Element Size Parameters section. Select the Maximum element size check box.

5 In the associated edit field, type 10 [um].
6 In the Model Builder window, right-click Mesh I and choose Free Tetrahedral.
7 Right-click Mesh I and choose Build All.

STUDY I

## Solver I

I On the Study toolbar, click Show Default Solver.
2 In the Model Builder window, expand the Study I>Solver Configurations node.
3 In the Model Builder window, expand the Solver I node, then click Dependent Variables I.

4 In the Dependent Variables settings window, locate the Scaling section.
5 From the Method list, choose None.
6 On the Study toolbar, click Compute.

## RESULTS

## Velocity (nitf)

The first default plot shows the velocity magnitude on slices.

## Temperature (nitf)

The second default plot shows the temperature on channel inner surfaces.
To reproduce Figure 3, proceed as follows:

## 3D Plot Group 3

I On the Home toolbar, click Add Plot Group and choose 3D Plot Group.
2 In the Model Builder window, under Results right-click 3D Plot Group 3 and choose Isosurface.

3 In the Isosurface settings window, locate the Expression section.
4 Verify that the Temperature ( $\mathbf{T}$ ) is defined as default expression.
5 In the Levels edit field, type range (301,2,329).
6 Locate the Coloring and Style section. From the Color table list, choose ThermalLight.
7 In the Model Builder window, right-click 3D Plot Group 3 and choose Streamline.
8 In the Streamline settings window, locate the Expression section.
9 Click Conductive heat flux (nitf.dfluxx,...,nitf.dfluxz) in the upper-right corner of the section. Locate the Streamline Positioning section. From the Positioning list, choose Start point controlled.
10 Locate the Coloring and Style section. From the Line type list, choose Tube.
II Select the Radius scale factor check box.
I2 On the 3D plot group toolbar, click Plot.
I3 Right-click 3D Plot Group 3 and choose Rename.
14 Go to the Rename 3D Plot Group dialog box and type Temperature isosurfaces and Conductive heat flux streamlines in the New name edit field.

I5 Click OK.

